

Optimization of vehicle handling performance by increasing the ARB effectiveness

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BY

Dr. A K Jindal, M.G. Belsare and T. M. Arun Prakash

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- Vehicle Specifications
- Suspension System Configuration
- Subjective Appraisal and Problem Statement
- Concept Evaluation
- Concept Finalization
- Modified Component List and System Configuration
- Design Of Experiments
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GRANDE MK II
More than meets the eye

TATA MOTORS



Vehicle Specifications

Sr.	System	Details
1	Engine	2.2 L 16 Valve DOHC DICOR and 2179 CC
2	Steering	RCBT steering gear box with power steering and collapsible with tilt steering column
3	Suspension	
	Front	Double wishbone type with coil spring
	Rear	Solid Axle (Hotchkiss drive) with parabolic leaf spring
4	Tyres	235 / 70 R 16 Tubeless tires
5	Brake	
	Actuation	Hydraulic brakes with Vacuum Assisted
	Foundation	Ventilated disc with twin pot caliper at front and Auto adjusted drum brake on rear

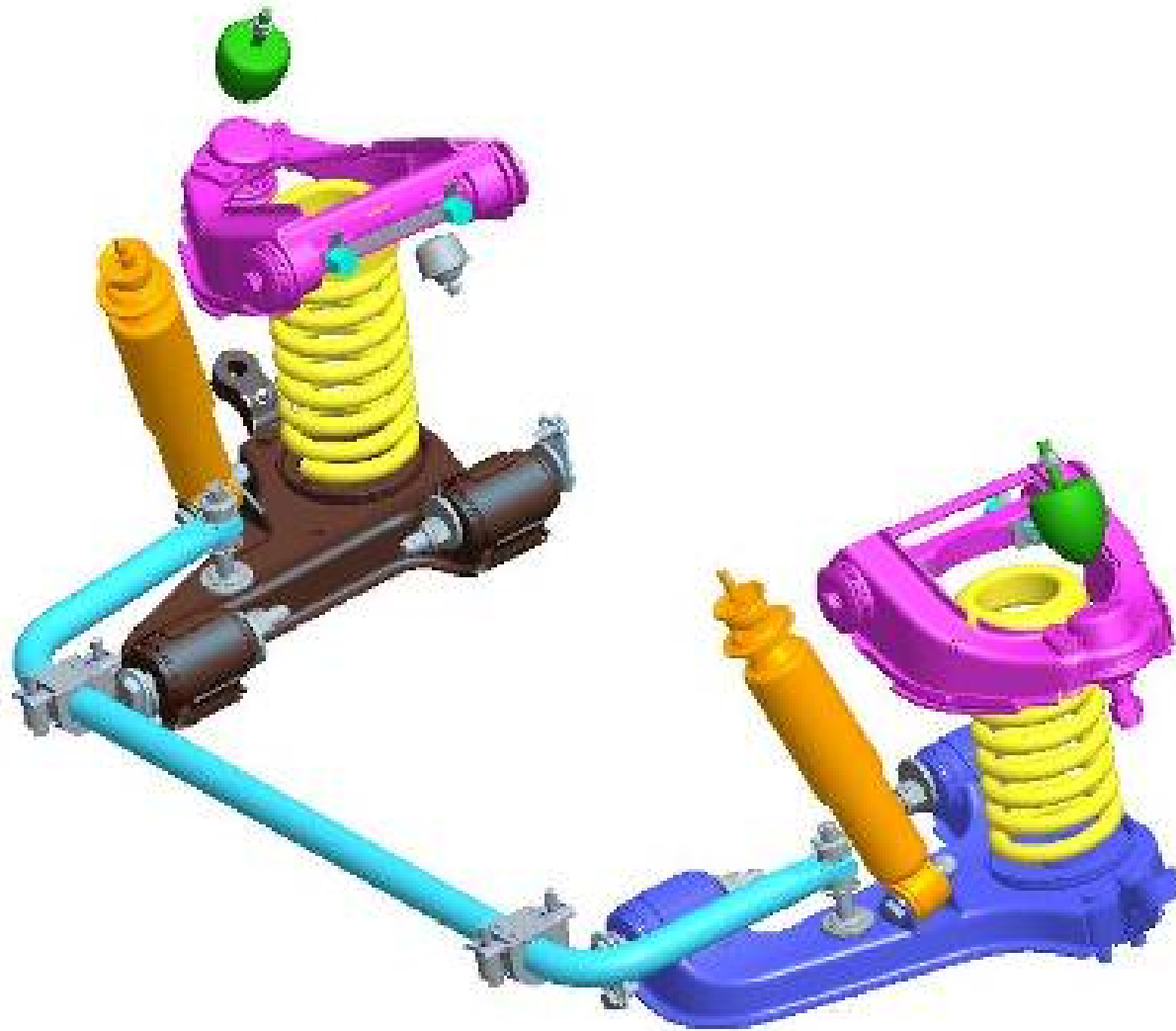
Chassis Dimension

SI No	Details	Unit	Front	Rear
1	Unladen weight	Kg	1000	950
2	Track	mm	1496	1490
3	Overall length	mm	4421	
4	Max Width	mm	1780	
5	Overall Height	mm	1940	
6	Ground Clearance	mm	180	
7	Turing Radius	m	5.25	
8	Wheel Base	mm	2550	

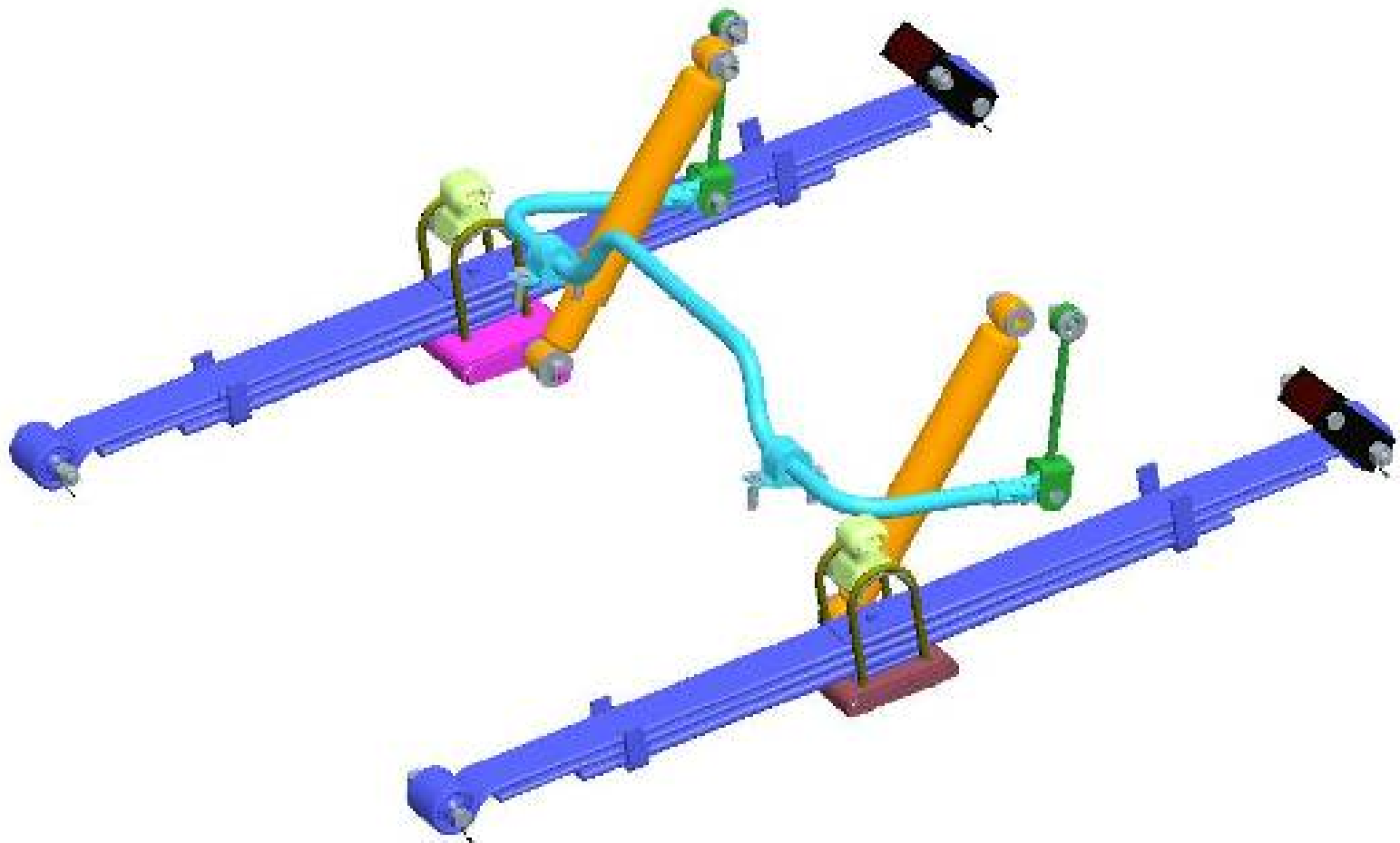
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Front Suspension View



Rear Suspension View



Demands of SUV from the suspension system

- To minimize the low frequency motions of the sprung masses (i.e.) bouncing, rolling, pitching ground 1 Hz.
- To avoid compiling between suspension resonances and chassis vibrations.
- rolling of the body ↓
- corner stability ↑
- braking stability ↑

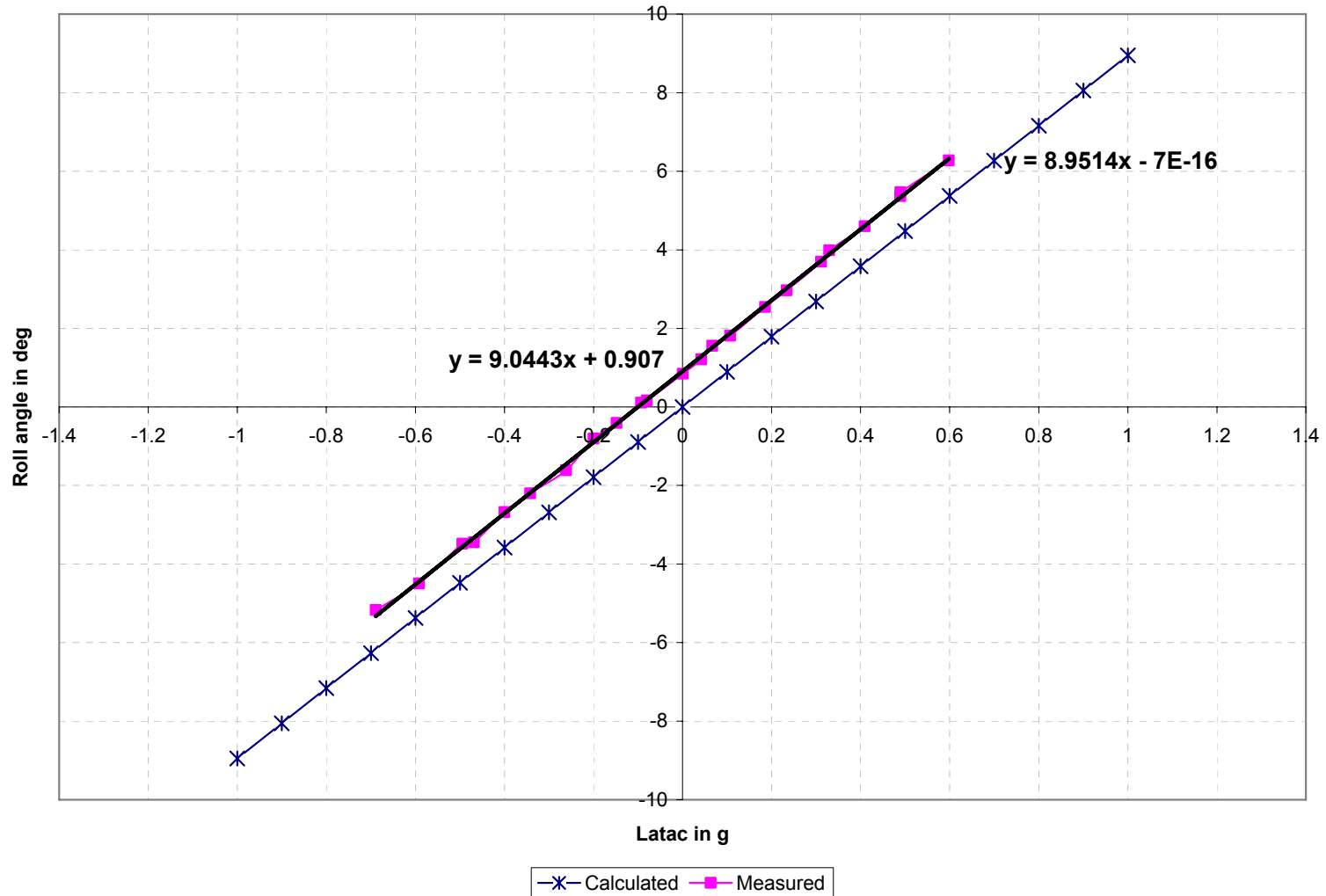
Demands of SUV from the suspension system

- force at the steering wheel ↓
- effects due to loading variations between one driver & full load ↓
- Strong package boundaries ↑
- ground clearance ↑

Suspension System Configuration

Sl. No	Description	Unit	Front	Rear
1	Ride Frequency	Hz	1.5	1.6
2	Unladen Ride Travel	mm	45	84
3	Roll Center Height	mm	26	242
4	CG from the ground	mm	725	
5	ARB Dia	mm	32	22
6	ARB Stiffness	Kg m/deg	177	26
7	ARB Effectiveness at the wheels	Kg m/deg	24	13
8	Total Roll stiffness of the vehicle	Kg m/deg	97	61
9	% of ARB Contribution in the roll stiffness	%	16.5	16.4

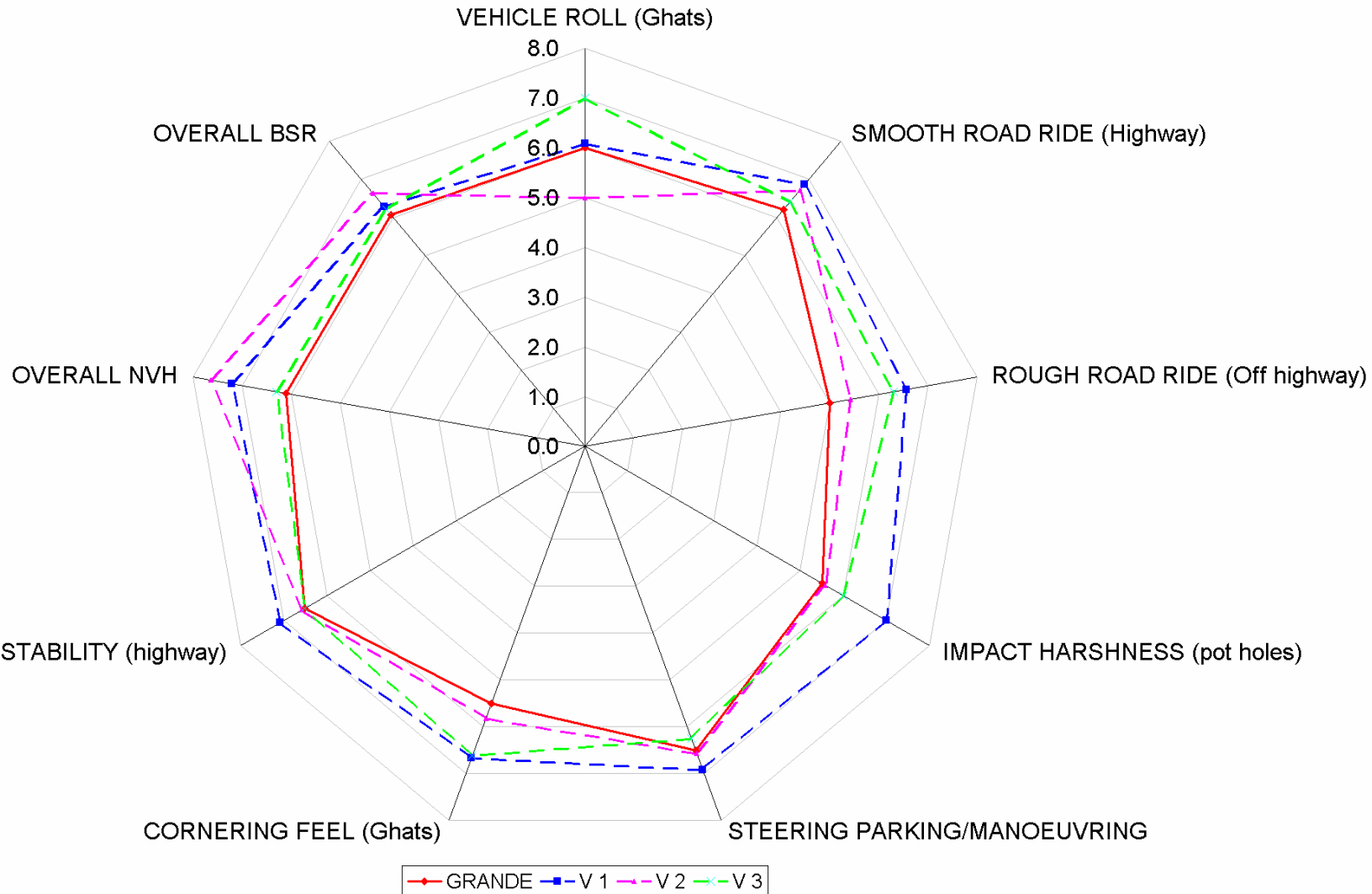
Roll Gradient Measured and Calculated



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Subjective Appraisal of GRANDE MK-I with Benchmarks



Problem Statement

- Roll of the vehicle is high when compared with benchmark vehicle
- Ride is Harsh
- Pitching
- Poor Cornering feel at the Hilly regions

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Concept Evaluation

- Roll gradient of the vehicle to be reduced without changing the suspension hard points
- Options to reduce the roll gradient of the vehicle
 1. front roll stiffness ↑
 2. rear roll stiffness ↑

Design options

- Options for Roll stiffness ↑
 - 1) spring stiffness ↑
 - 2) diameter of the ARB ↑
 - 3) spring track ↑
 - 4) Change the geometry of the ARB

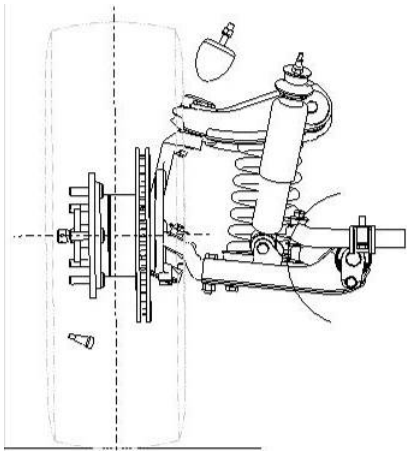
Design option Finalization

- 1) spring stiffness ↑ ----- Ride Comfort
- 2) diameter of the ARB ↑ ----- weight and effectiveness
- 3) spring track ↑ ----- Packaging constraint and hard point change
- 4) Change the geometry of the ARB

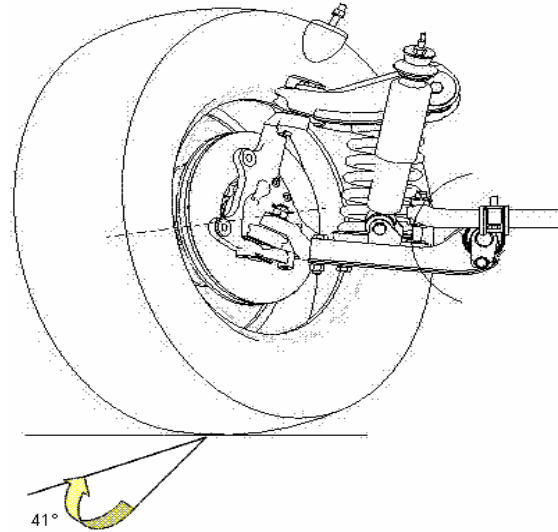
Existing ARB Details

- Mounted on the wishbone --single degree of freedom
- End link is conventional (Bush- Bush)
- effectiveness is less
- 16% of its total roll stiffness

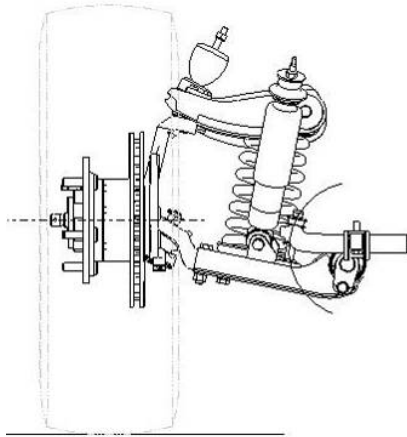
Existing ARB movement in Different Conditions



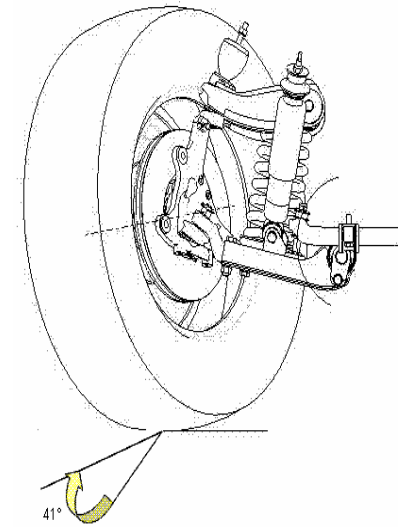
Unladen Straight Ahead



Unladen Full Inner turn

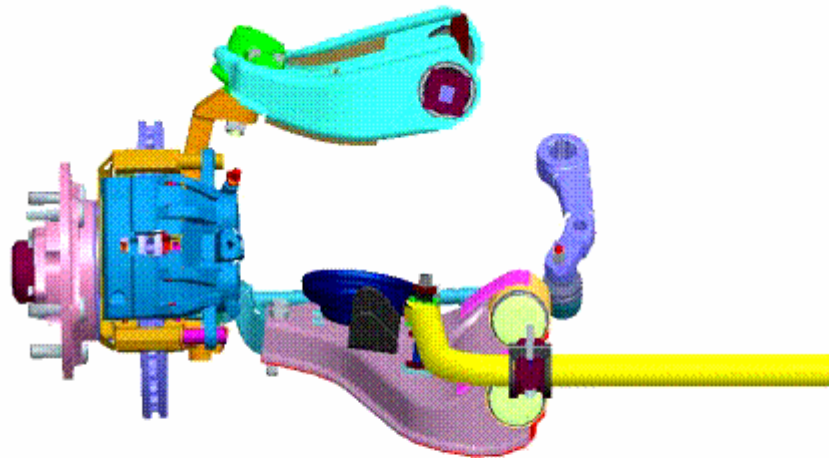


Bump Straight Ahead



Bump Full Inner Turn

Steering and Suspension Motion Clip



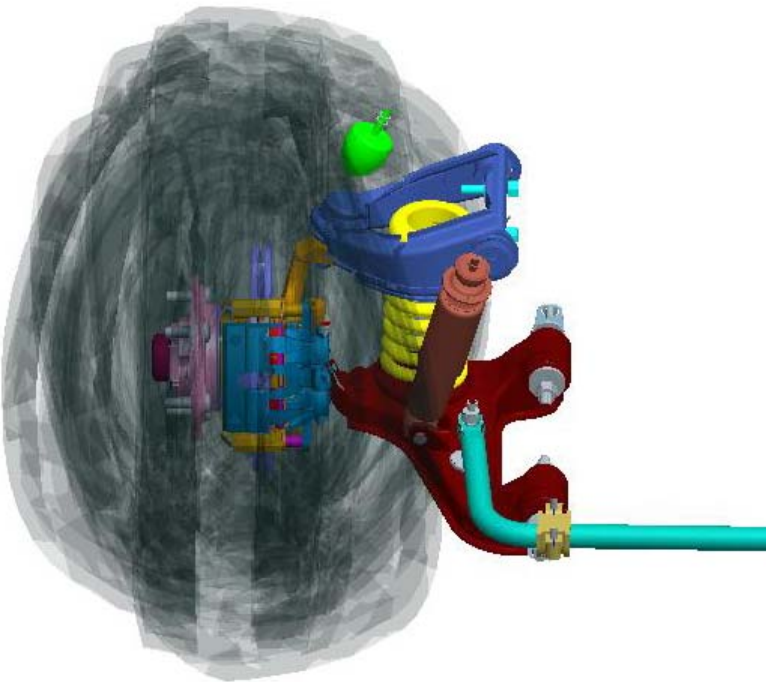
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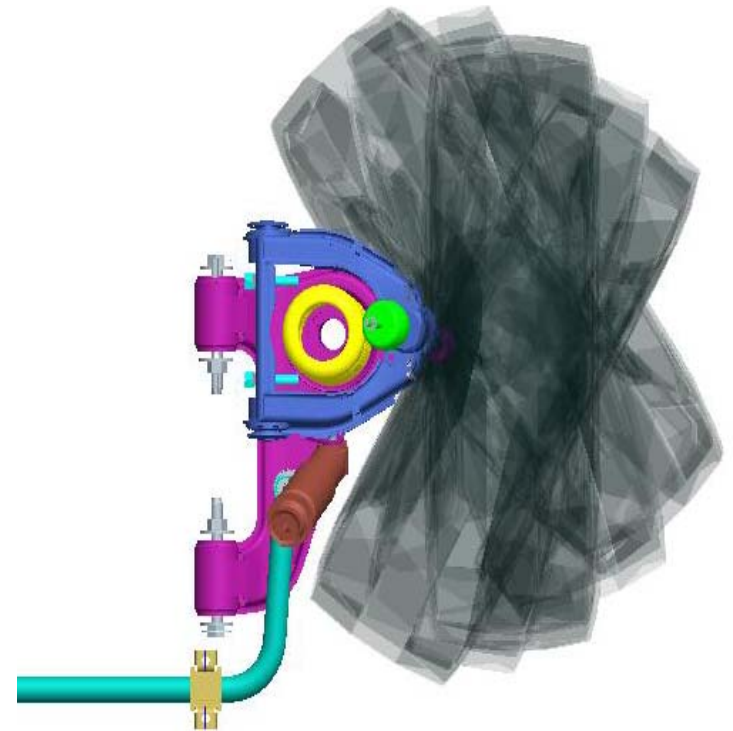
Concept Finalization

- Relocation is critical due to the wheel envelope clearance.

Rotated Front View



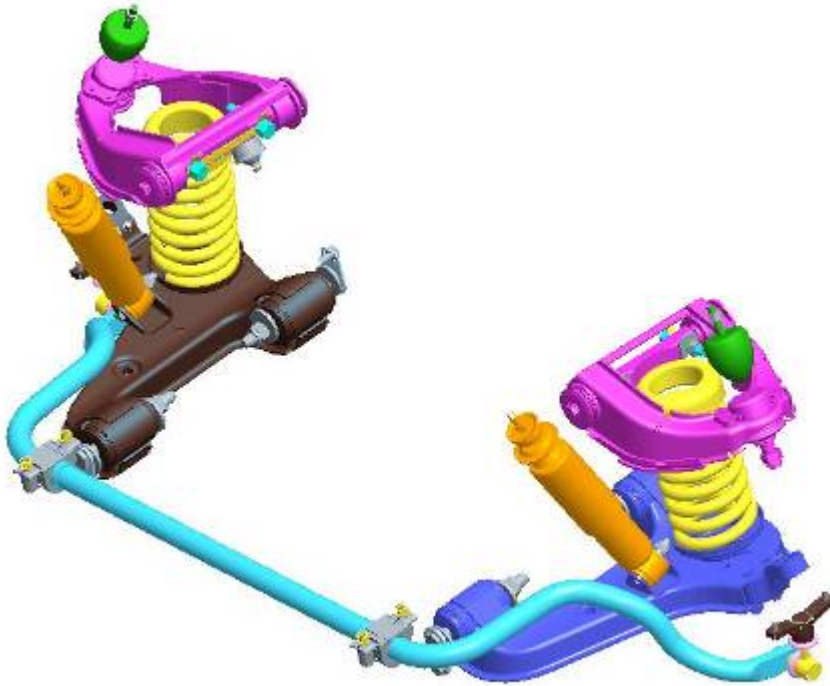
Top View



Cont...

- Joining the ARB in the stub Axle which has two degrees of the freedom
 - vertical motion (Up and Down for the bump and the rebound)
 - the rotational movement along the KPI
- Mounting location of the ARB is optimized on the bottom of the stub Axle along the KPI Axis
- New Ball joint designed

Modified ARB with Suspension System

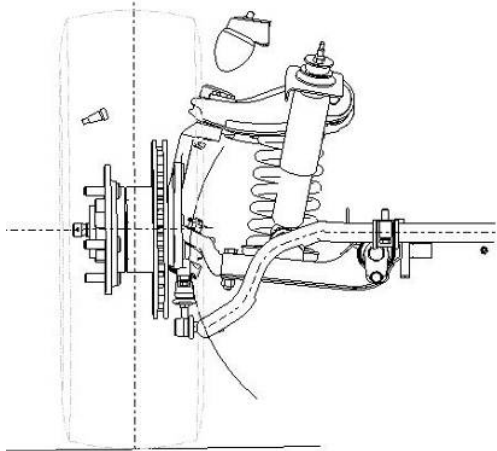


Ball Joint

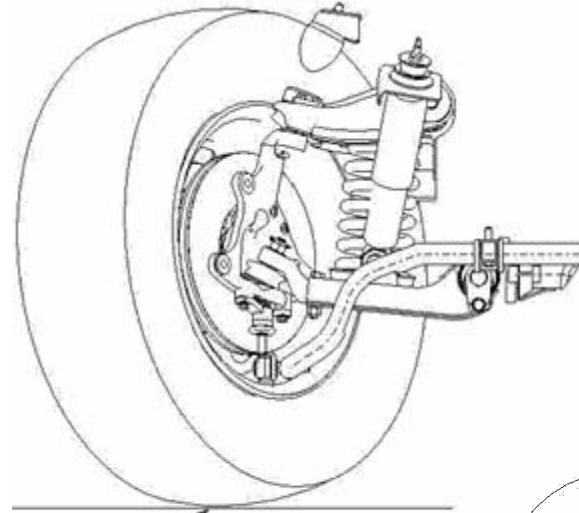
Suspension System with the Wheel envelope



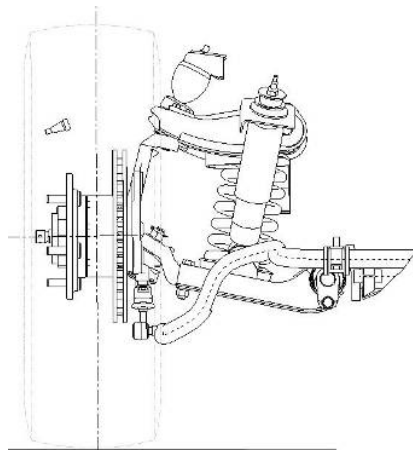
ARB Movement in Different Conditions



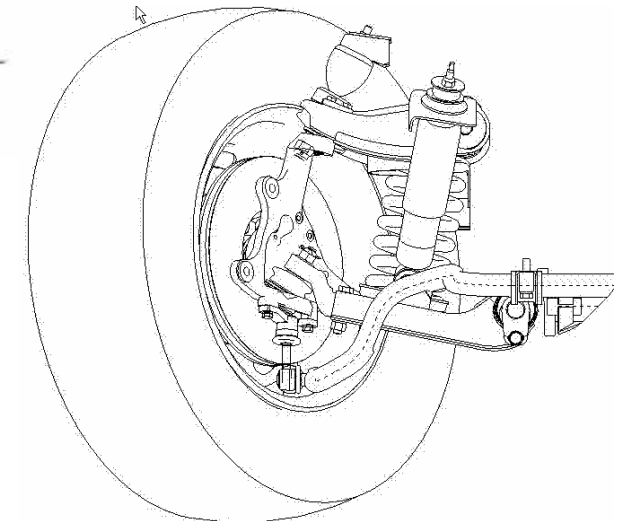
Unladen Straight Ahead



Unladen Full Inner turn

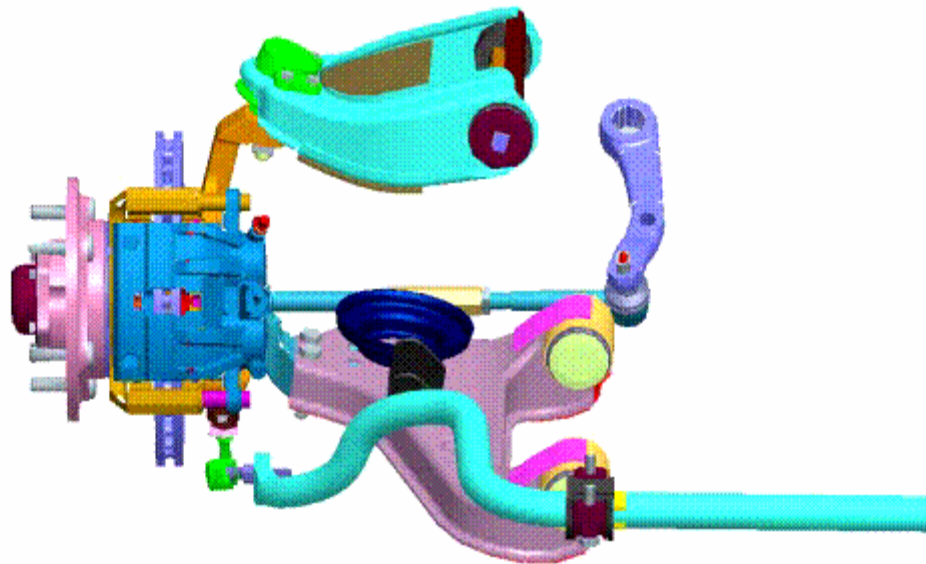


Bump Straight Ahead



Bump Full Inner Turn

Steering and Suspension Motion Clip for Modified ARB



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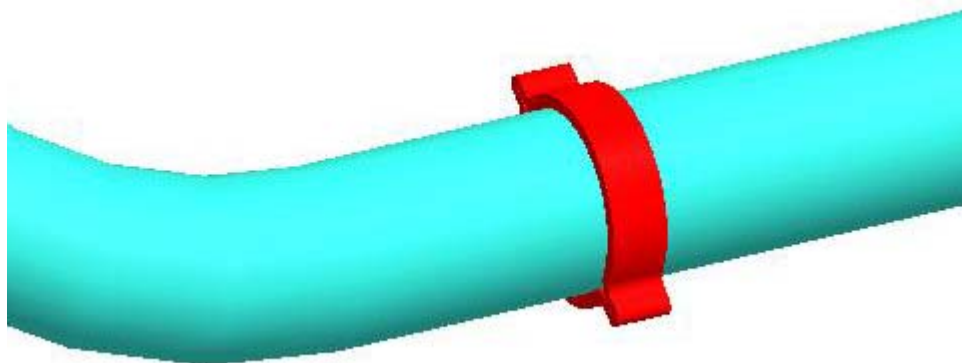
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Modified Component List

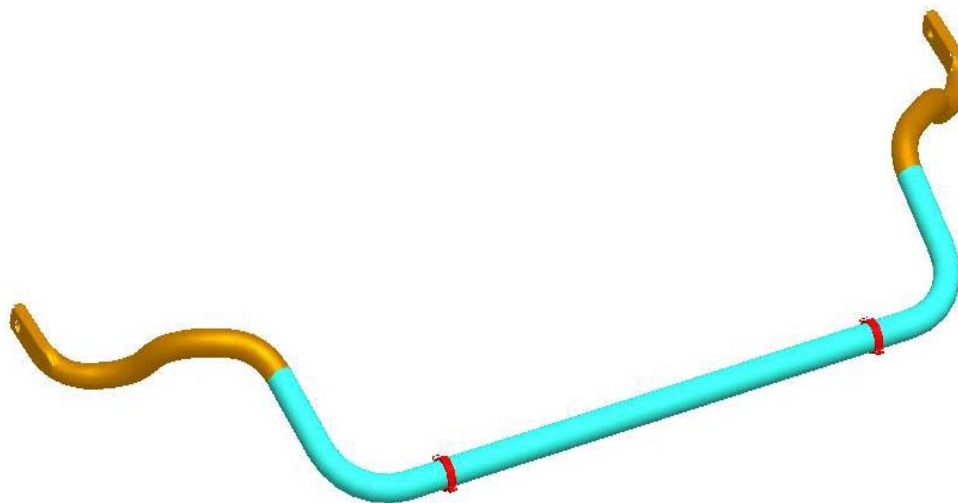
1. Stub Axle
2. ARB
3. ARB Ball Joint
4. Lower wishbone
5. Shock Absorber mounting bracket Top and Bottom

Implementation Difficulties

- Crimping



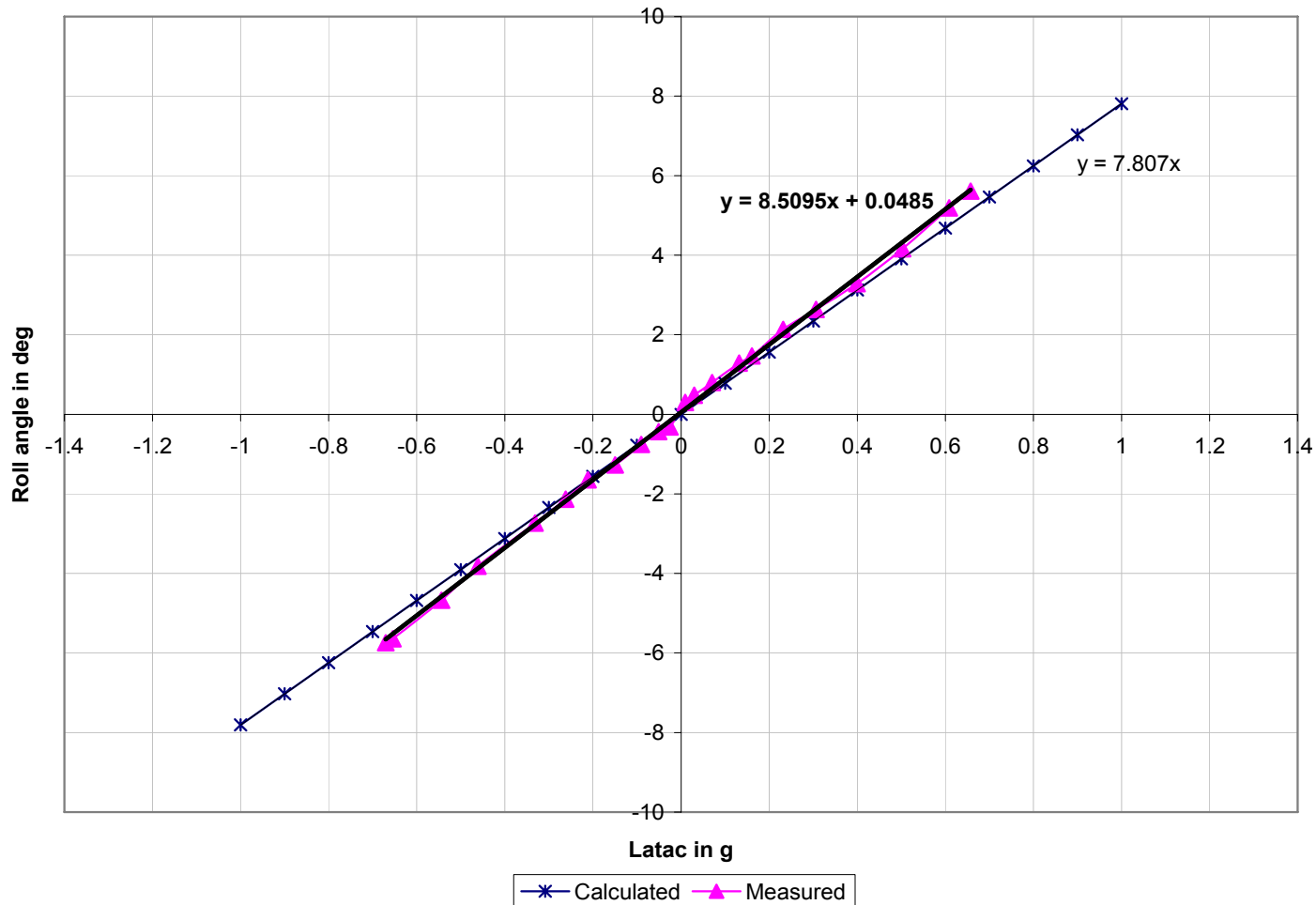
- Tapered ARB



Modified Suspension System Configuration

Sl. No	Description	Unit	Front Existing	Front Modified	Rear
1	Ride Frequency	Hz	1.5	1.5	1.6
2	Unladen Ride Travel	mm	45	45	84
3	Roll Center Height	mm	26	26	242
4	CG from the ground	mm	725	725	
5	ARB Dia	mm	32	30	22
6	Bar Rate	Kg m/deg	177	78	26
7	ARB Effectiveness at the wheels	Kg m/deg	24	57	13
8	Total Roll stiffness of the vehicle	Kg m/deg	97	117	61
9	% of ARB Contribution in the roll stiffness	%	16.5	30.8	16.4

Roll angle Vs Latac with wider front 30mm dia ARB and rear 22 mm dia ARB



Significance of Achievements

- Roll stiffness ↑ from 97 Kgm/ deg to 117 Kgm/deg
- 30.8 % of the total roll stiffness
- Roll of the vehicle ↓

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Criteria for optimizations

- Balance between front and rear roll stiffness
- Higher overall levels of roll stiffness result in reduced body roll angles.
- To increase the effectiveness and to balance front and rear roll stiffness the following are the options

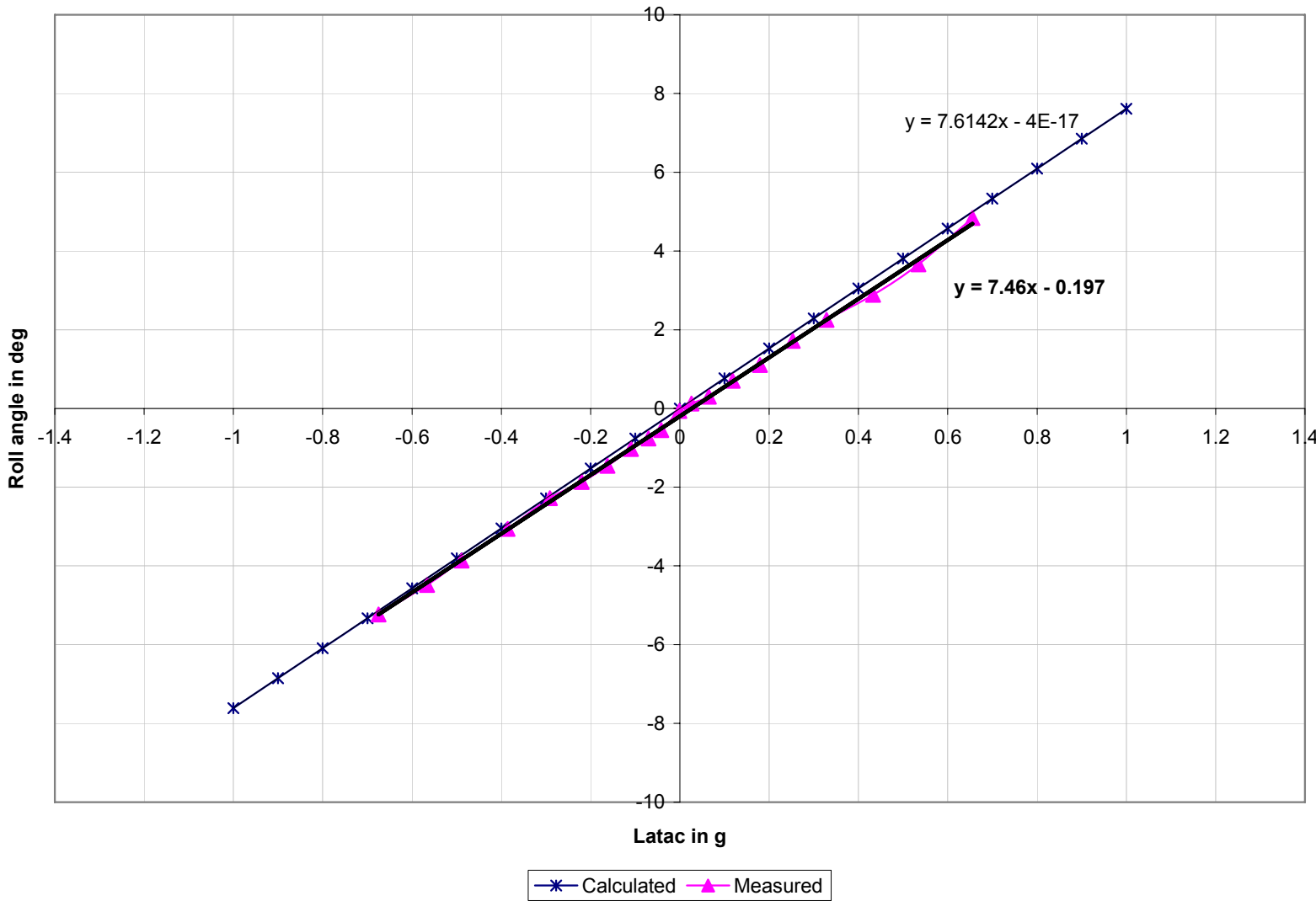
DOE

Options	A	B	C	D	E
Coil Spring Stiffness in kg/mm	14.4	14.4	14.4	14.4	12.8
Front ARB Dia in mm	30	32	32	32	32
Leaf Spring stiffness in kg/mm	4.7 / 7.8	4.7 / 7.8	4.7 / 7.8	4.7 / 7.8	4 / 7.5
Rear ARB Dia in mm	24	22	24	28	24

(A) Front ARB 30 mm diameter and Rear 24 mm diameter

Sl. No	Description	Unit	Front	Rear
1	ARB Dia	mm	30	24
2	Bar Rate	Kg m/deg	78	37
3	ARB Effectiveness at the wheels	Kg m/deg	57	18
4	Total Roll stiffness of the vehicle	Kg m/deg	117	65
5	% of ARB Contribution in the roll stiffness	%	30.80	21.50

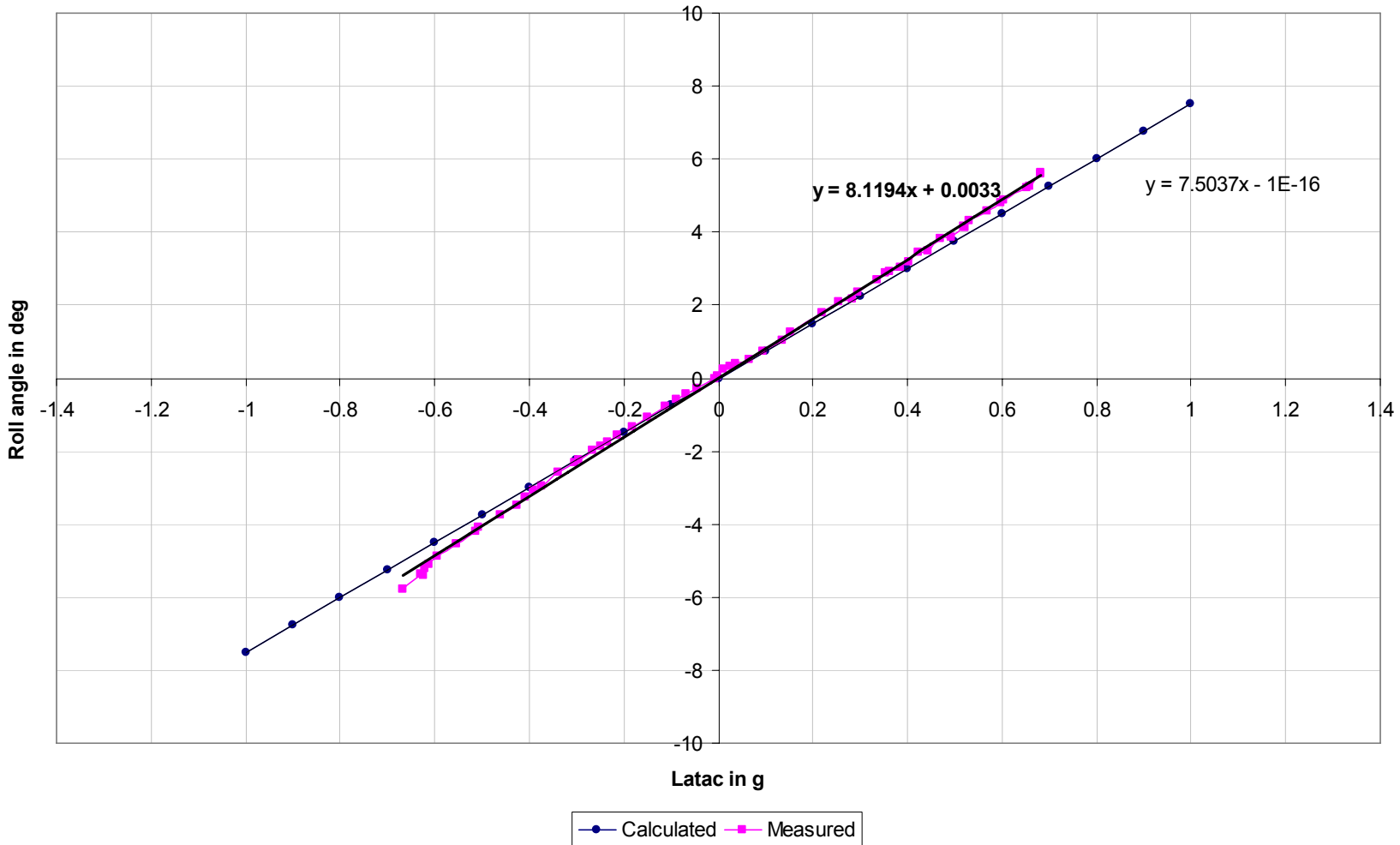
(A) Front ARB 30 mm diameter and Rear 24 mm diameter



(E) Front ARB 32 mm diameter and Rear 24 mm diameter with modified Spring Characteristics

Sl. No	Description	Unit	Front	Rear
1	Ride Frequency	Hz	1.45	1.51
2	Unladen Ride Travel	mm	45	89
3	Roll Center Height	mm	26	246
4	CG from the ground	mm	725	
5	ARB Dia	mm	32	24
6	Bar Rate	Kg m/deg	101	37
7	ARB Effectiveness at the wheels	Kg m/deg	74	18
8	Total Roll stiffness of the vehicle	Kg m/deg	123	58
9	% of ARB Contribution in the roll stiffness	%	38.2	24.13

(E) Front ARB 32 mm diameter and Rear 24 mm diameter with modified Spring Characteristics



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Summary of DOE

Sr.	Front Suspension				Rear Suspension		Calculated	Measured	
	Coil Spring Stiffness in kg/mm	ARB Dia in mm	ARB end Mtg	ARB Link Arrangement	Leaf Spring Stiffness in Kg/mm	ARB Dia in mm	Roll gradient in deg/g	Roll gradient in deg/g	U/S in deg/g
Base	14.4	32	Wishbone	Bush- Bush	4.7 / 7.8	22	8.95	9.04	3.5
modi	14.4	30	Stub Axle	Ball- Ball	4.7 / 7.8	22	7.81	8.51	5.55
A	14.4	30	Stub Axle	Ball- Ball	4.7 / 7.8	24	7.61	7.46	4.74
B	14.4	32	Stub Axle	Ball- Ball	4.7 / 7.8	22	7.36	7.01	3.86
C	14.4	32	Stub Axle	Ball- Ball	4.7 / 7.8	24	7.19	7.16	4.17
D	14.4	32	Stub Axle	Ball- Ball	4.7 / 7.8	28	7.06	7.33	2.91
E	12.8	32	Stub Axle	Ball- Ball	4.0 / 7.5	24	7.5	8.12	3.8

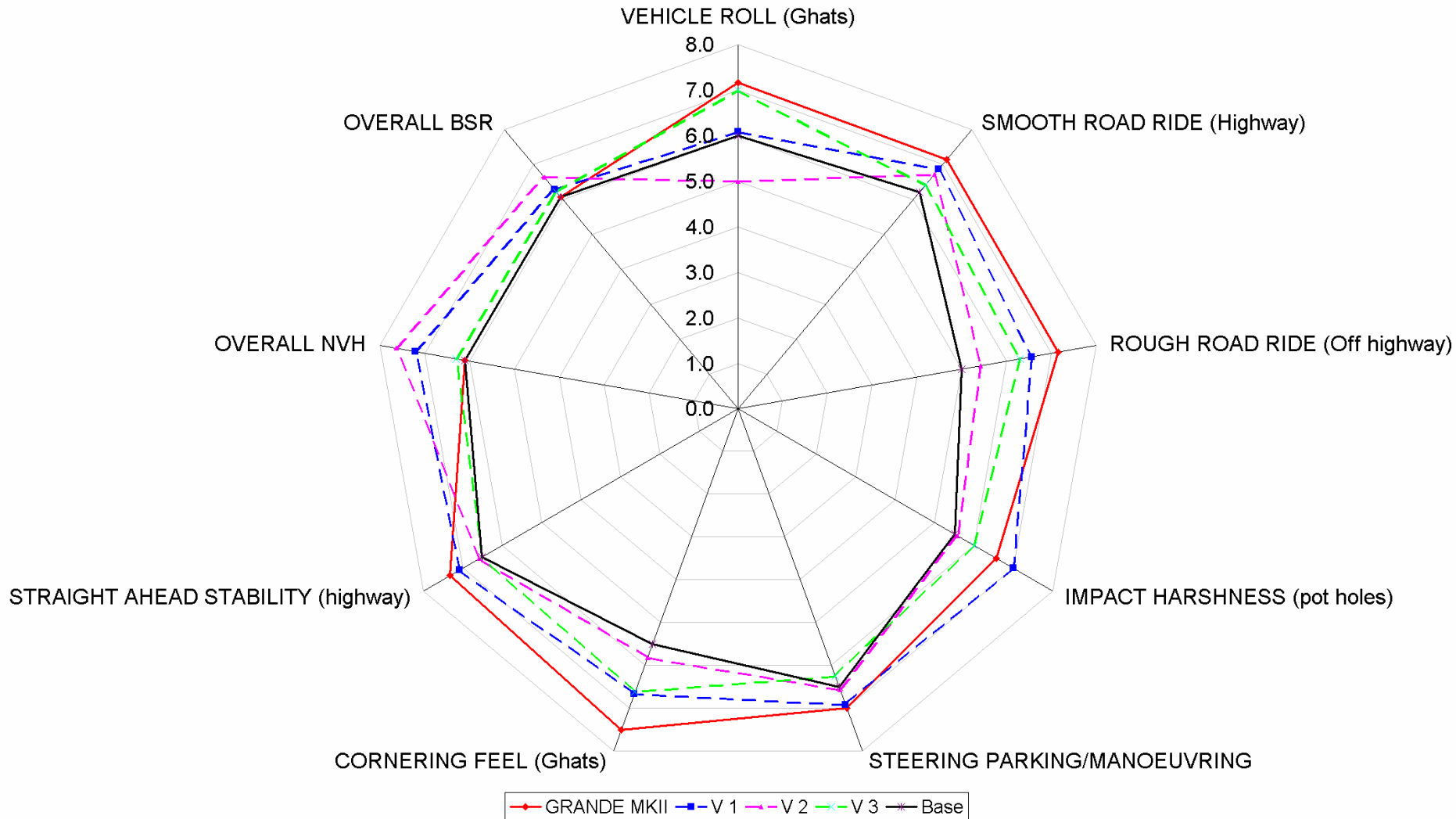
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Subjective Evaluation

- always the final judgment
 - it is practical
 - account many different conditions of the vehicle use.
- Parameters evaluated subjectively for the DOE configurations.
 - Straight running Stability
 - Lane change maneuverability
 - Cornering Stability
 - Steering effort.
 - Ride Comfort
- Option (E) holds good in the subjective evaluation for the above parameters.

Subjective Appraisal of GRANDE MK-II with Benchmarks



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Conclusions

- By joining the ARB along the KPI Axis
 - Steering Kinematics is isolated from the ARB motion path
 - Articulation of the Ball joint is optimized- (No side load)
- The effectiveness of the ARB is increased
- By mounting the ARB end to Stub axle 100 % motion ratio is achieved.

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Discussions

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Thank You