



# Simulating Advanced Damping Technologies

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BWI Group acquired Suspension and Brakes business lines from Delphi in 2009

- 3 technical centers (Dayton, Paris, Krakow)
- 4 manufacturing plants (Krosno, Chihuahua, Luton, Fangshan (opens in 2011))

### TECHNICAL CENTER KRAKOW:

- Passive Dampers & Damper Modules Global Engineering Center
- Suspension Components Engineering (MR dampers, ASBS components)

Suspension headcount: 231

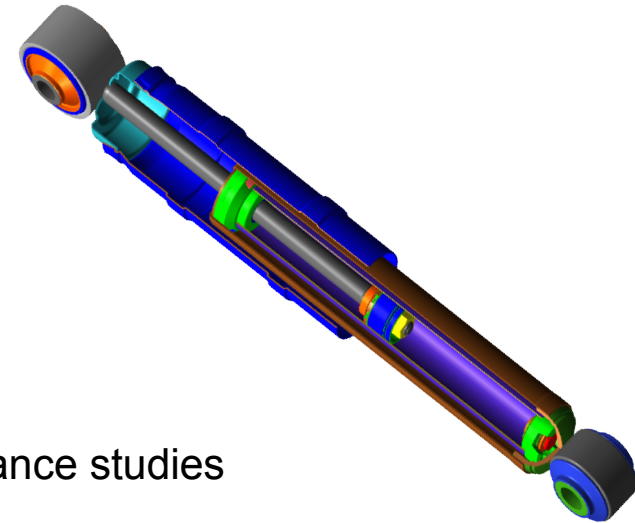
- Product, Manufacturing, Industrial Engineering
- CAD & CAE & Simulation
- Prototype Center
- Test & Validation Lab
- Metrology & Materials Lab
- Valving Lab & Ride Vans
- Supporting Functions (Purchasing, Finance, Logistics, Administration)



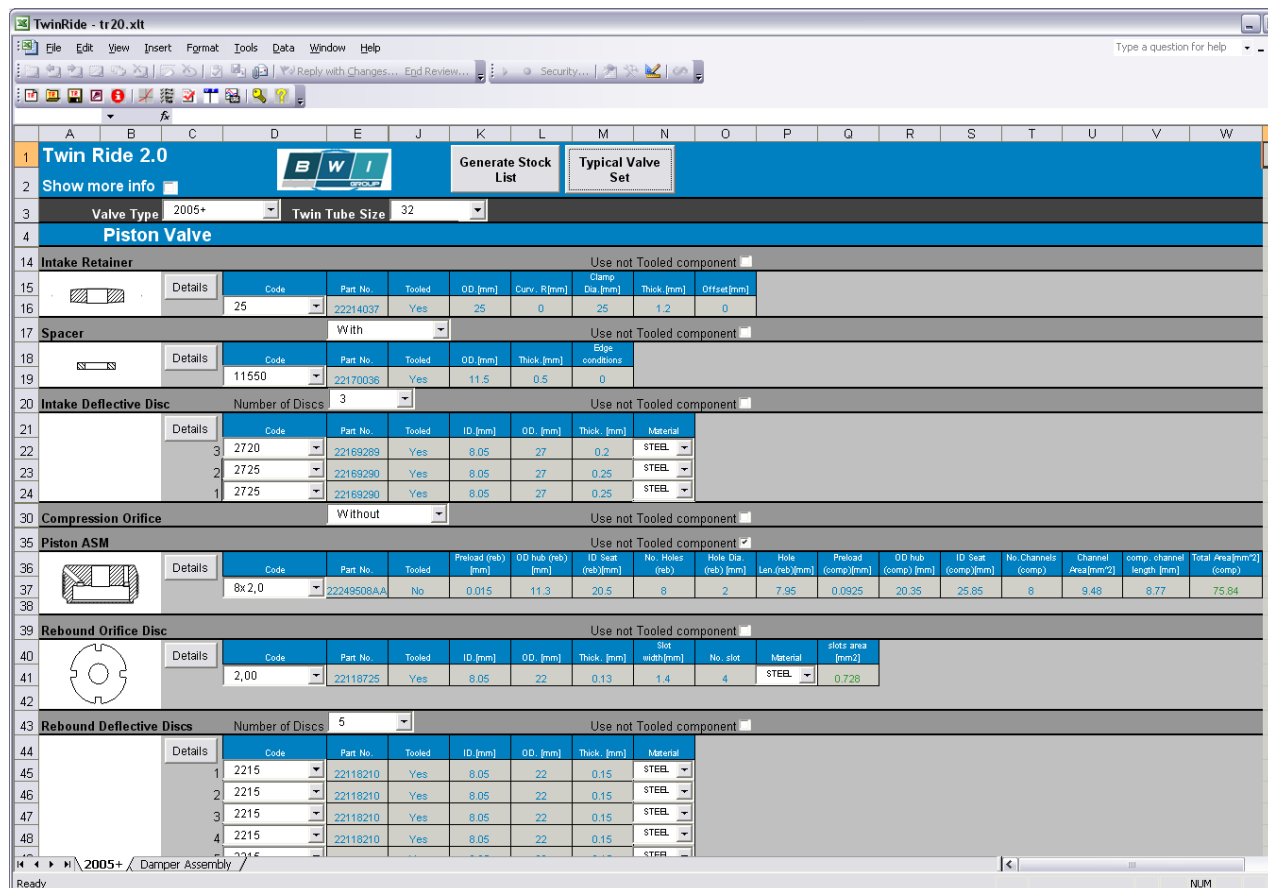


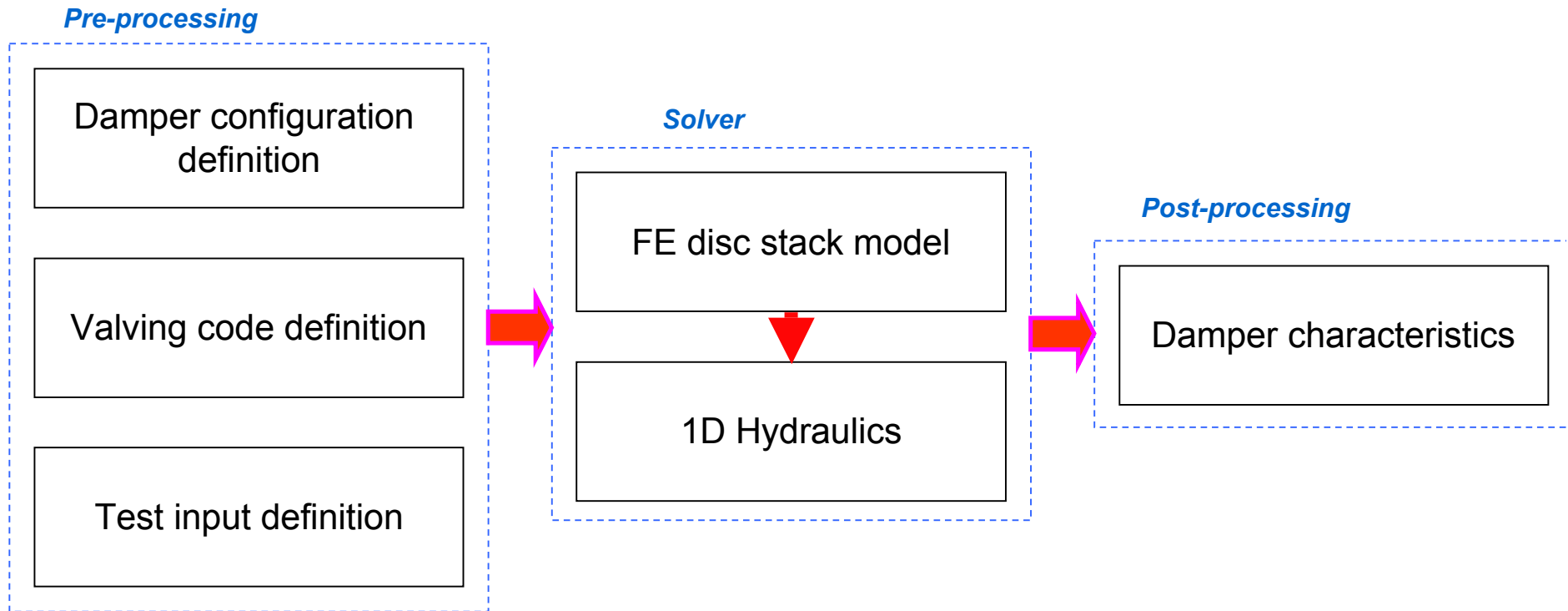
# Damper Modeling Tools @ BWI

- ◆ BWI has utilized damper modeling tools since 1990s
  - Support for nearly every passive or semi-active shock absorber configuration developed in-house
- ◆ Own codes that have been under continuous development and updated (improved) over the years
- ◆ Advantages
  - Accelerate damper & vehicle tuning, RFQ
  - Less iterations in the lab with the hardware
  - Valuable insight into damper physics
  - Analysis of damper design, incl. sensitivity and tolerance studies
  - Synthesis of damper design – design optimisation
  - Great learning tool for technicians and the engineering community
  - Valuable aid in valve code selection & proofing
  - Computer aided investigation, e.g. tracing assembly errors



- ◆ TwinRide is our standard tool for damper design and modeling
- ◆ Unified User Interface with a connection to a database of components



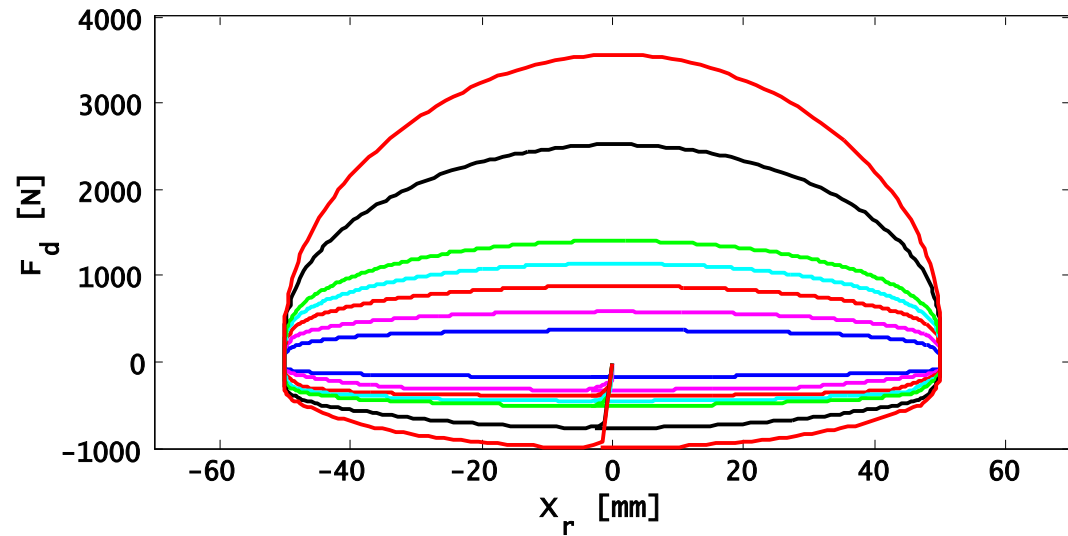
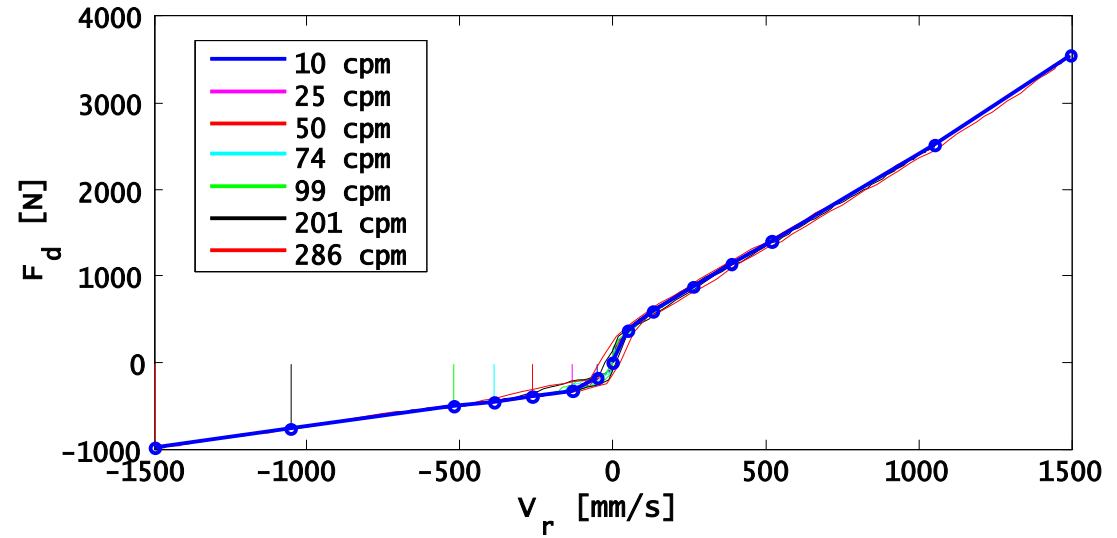
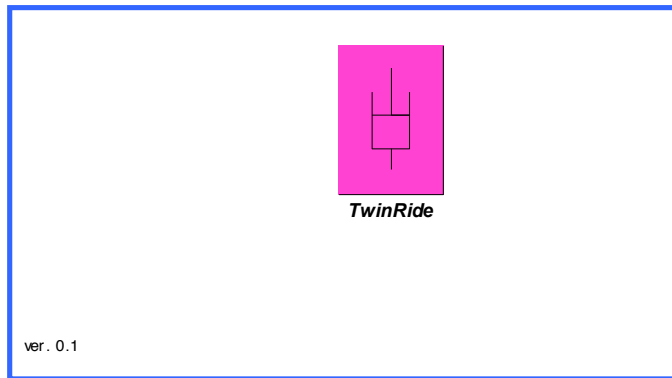


## Passive dampers only

## ◆ Modeling features

- Force vs. Velocity, Force vs. Displacement phase planes prediction (hysteresis)
- Steady-state Force vs. Velocity curve prediction
- Oil-gas mixture compressibility
- Fluid aeration
- Gas-to-liquid transformation equations
- Global cavitation model
- Tube flexibility (expansion with pressure)
- Leakage rate past piston and rod guide assemblies
- Piston, rod guide, spring seat friction
- Functional math-based piston and base valve models
- Deflection, stress characteristics for deflected disc assemblies
  
- Access to all important geometry variables via a database interface
- Unified user interface

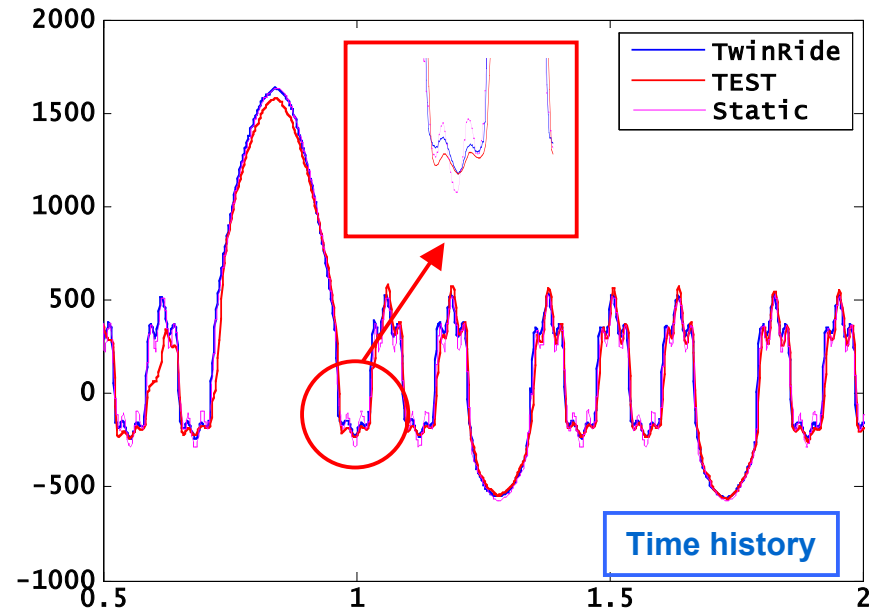
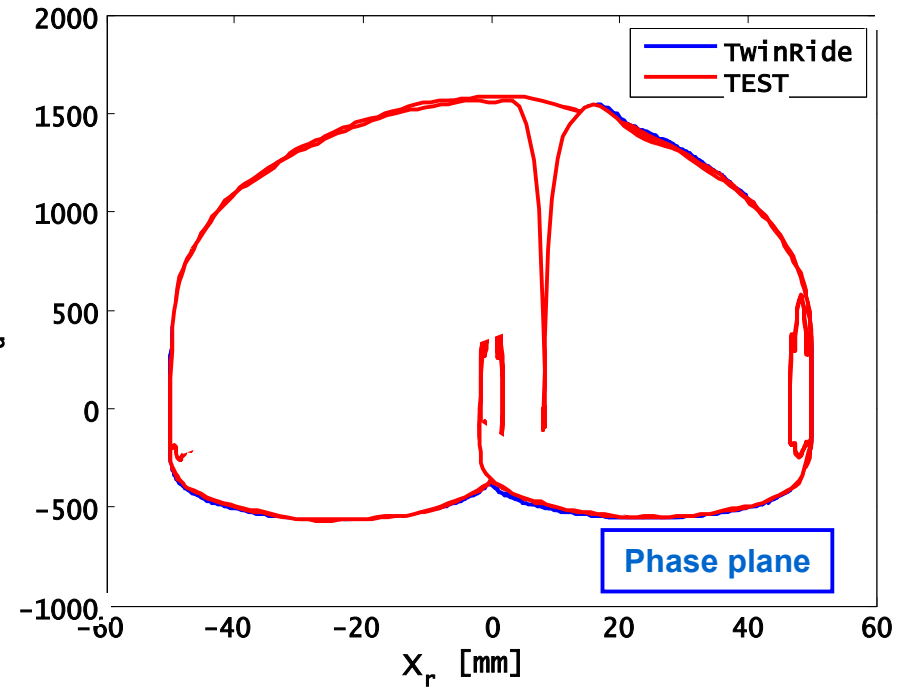
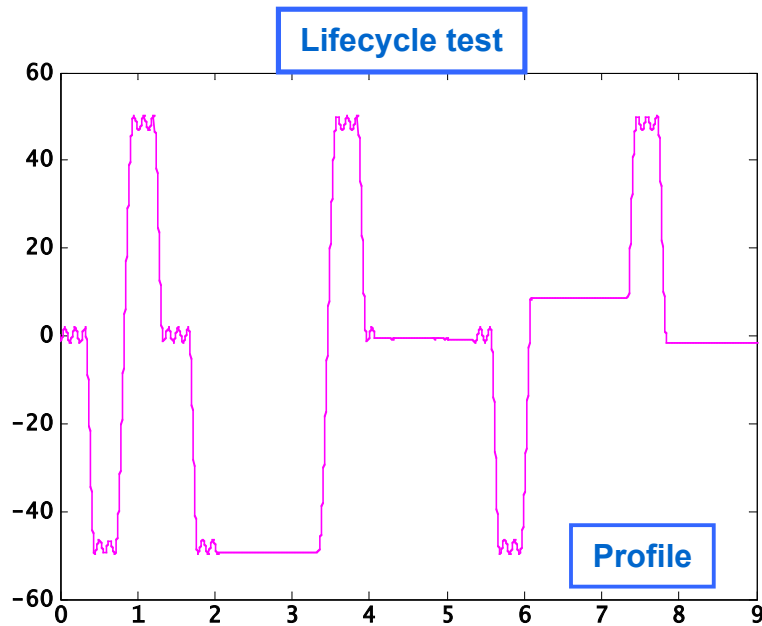
◆ Example #1  
 - Force vs. Velocity





◆ Example #2

- Response to arbitrary road profiles



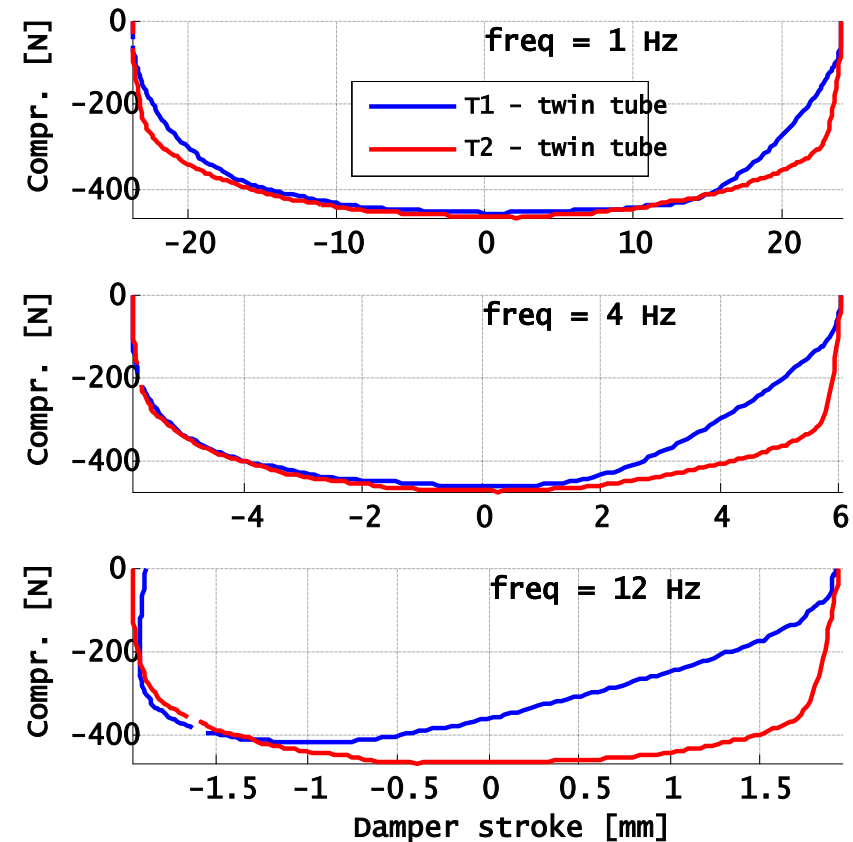
## ◆ Example #3

### – Vehicle tuning

- » 2 twin-tube shock absorbers
- » **Identical** steady-state F-V curves, **different** valve balance
- » **T1** – *soft* compression piston valve, *stiff* base valve
- » **T2** – *stiff* compression piston valve, *soft* base valve

- Quarter car results confirm the two codes have impact on vehicle ride
  - » E.g. better comfort, and road holding with **T2**

### Virtual tuning sessions



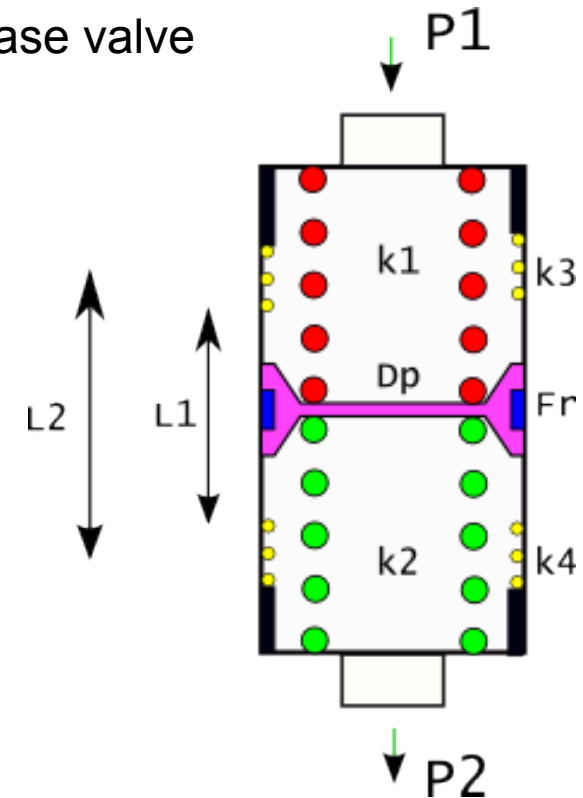


# Frequency Dependent Valving Model

- ◆ Background information
  - Current market trends have indicated a need for simple, inexpensive add-ons in ordinary shock absorbers
  - Notable examples: inertia valves, displacement-dependent valves, amplitude-selective valves (ASD), frequency-dependent valves (FDV)
- ◆ So-called Frequency-Dependent Valving (FDV) is a simple valving concept that seems to attract interest from car makers
- ◆ FDV is claimed to improve ride comfort & enhance vibration isolation w/o loss of handling and safety degradation
- ◆ FDV allows for lower damping force at higher frequencies yet maintains high damping forces at lower frequencies
- ◆ Typically, it operates in parallel to the main valve (piston) and/or base valve if used on a twin-tube shock absorber
- ◆ BWI has been developing FDV systems since 2002
  - 2 generations of FVD developed so far

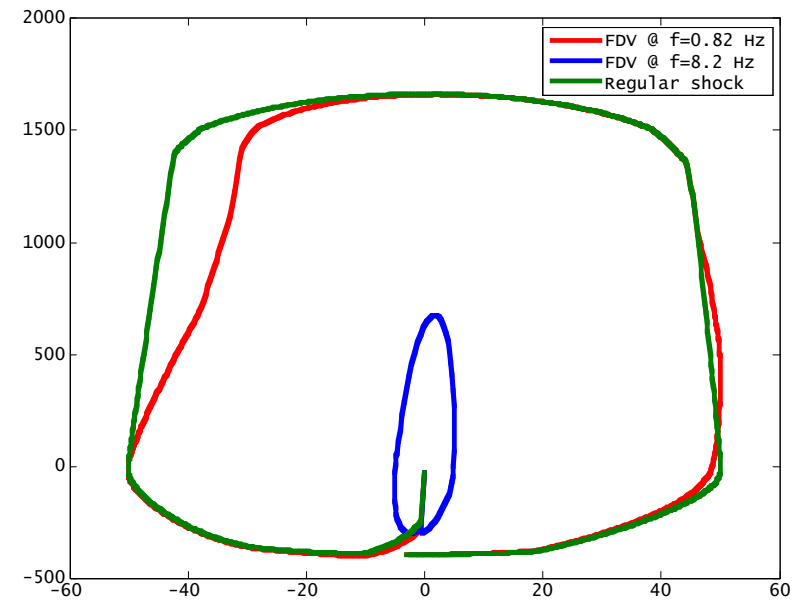
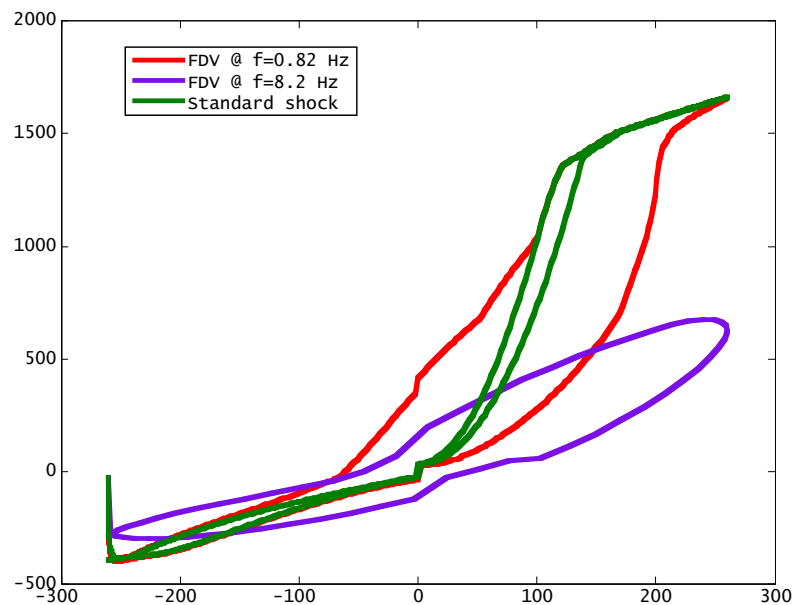
- ◆ FDV operates in parallel with the piston valve and/or base valve
- ◆ FDV Model parameters

- FDV piston diameter ( $D_{p1}$ ), massless piston
- Leading spring stiffness ( $k_1, k_2$ )
- Cushioning spring stiffness ( $k_3, k_4$ )
- Travel limits ( $L_1, L_2$ )
- Seal friction ( $F_r$ )
- Pressures ( $P_1, P_2$ )
- Flow rate ( $Q$ )
- Entry/exit restrictions



- ◆ By balancing forces acting on the FDV piston, we can derive the relationship between the flow rate through the FDV valve and the pressure drop across it
- ◆ FDV valve is then incorporated into the shock absorber model („damper-in-damper” approach)
- ◆ In case of a twin-tube damper, it results in a system of 5 (or 6) ODE equations that need to be solved simultaneously

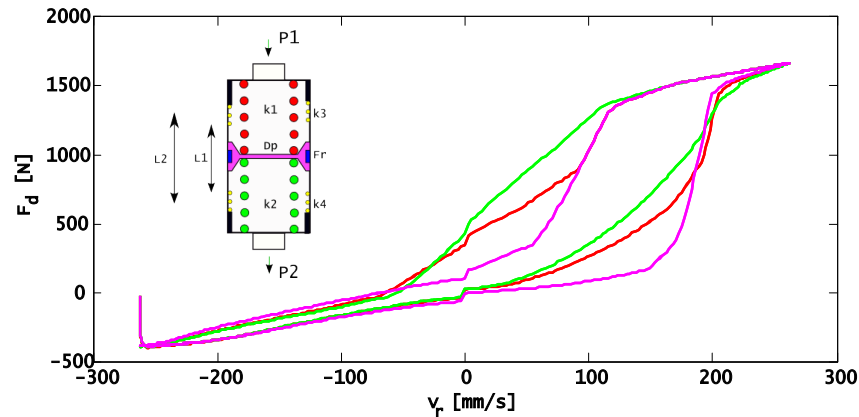
- ◆ It can be proven by mathematical analysis FDV performance is frequency-dependent
- ◆ Application of FDV in a shock absorber may have big impact on vehicle dynamic behavior
- ◆ A model is essential for understanding the FDV impact



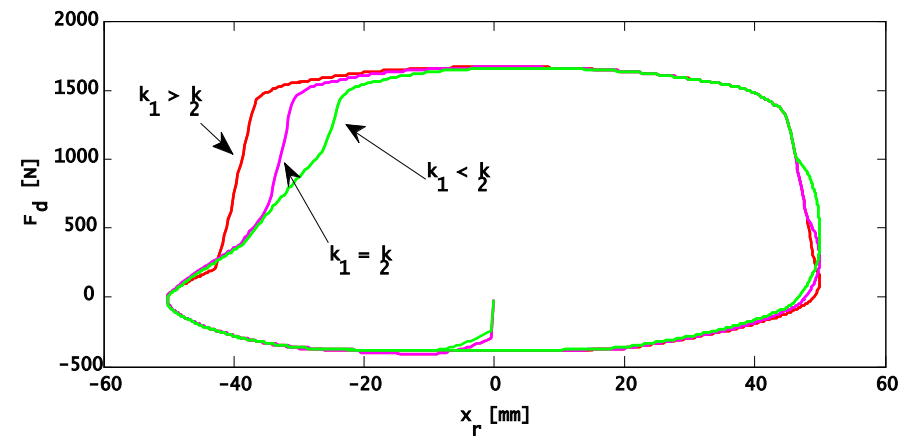
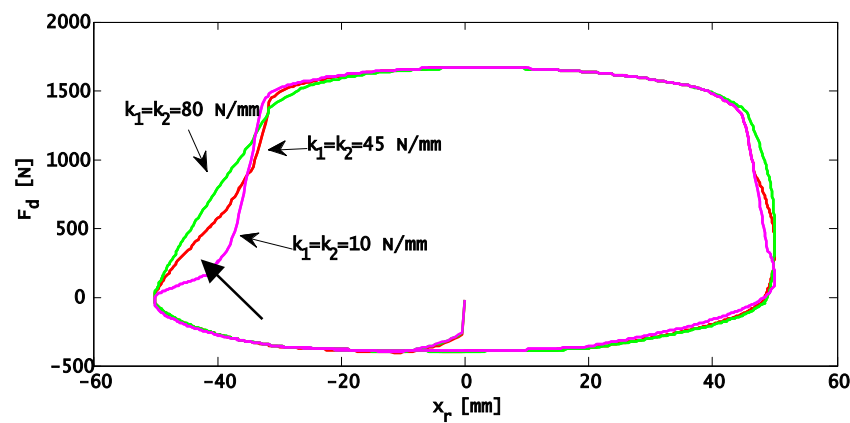
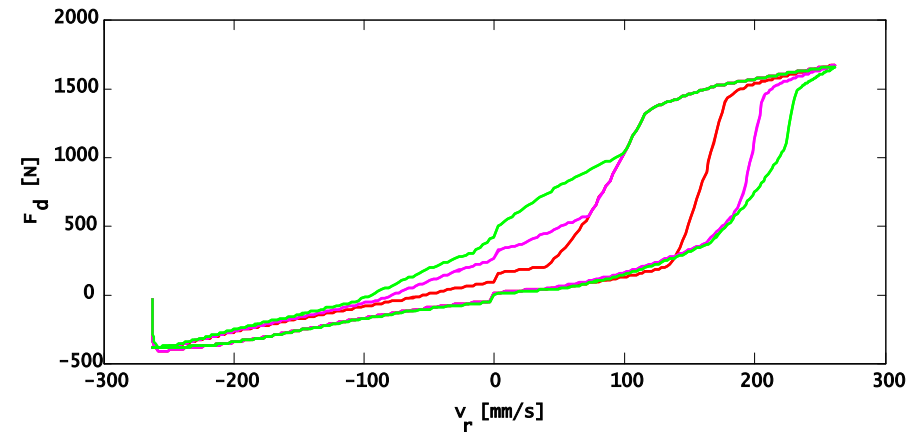
## Twintube shock absorber, FDV valve

## ◆ Simulation results – leading springs ( $k_1, k_2$ )

### Leading springs ratio change

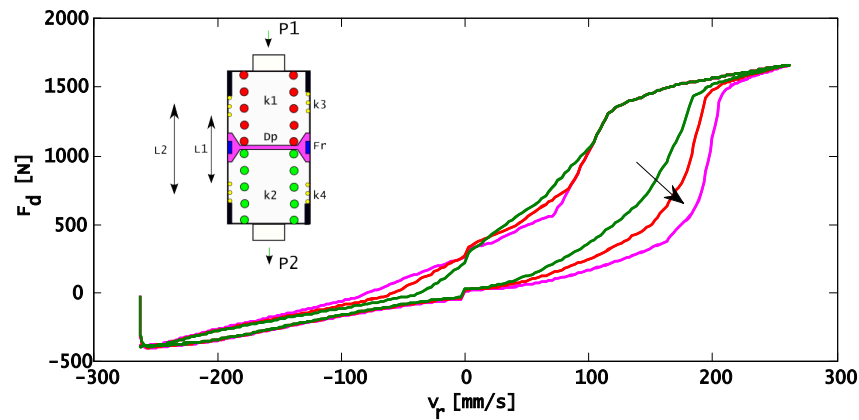


### Uneven leading springs ( $k_1 \neq k_2$ )

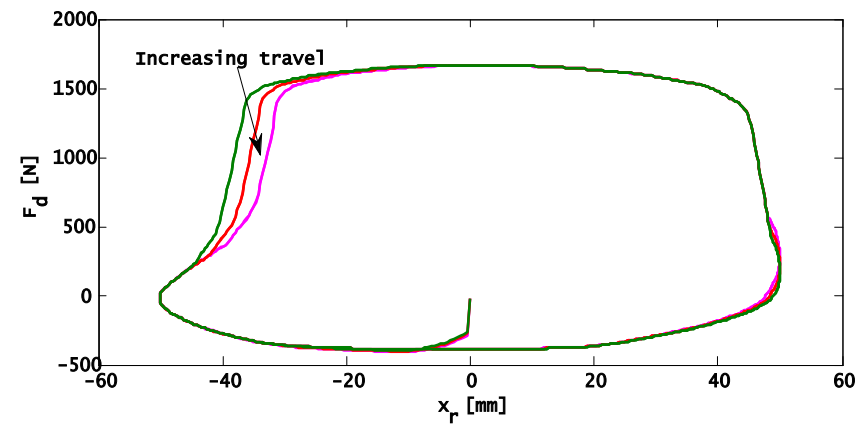
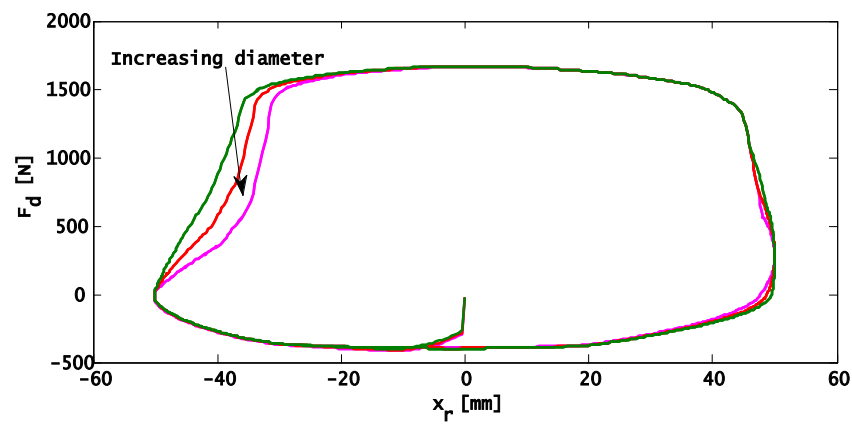
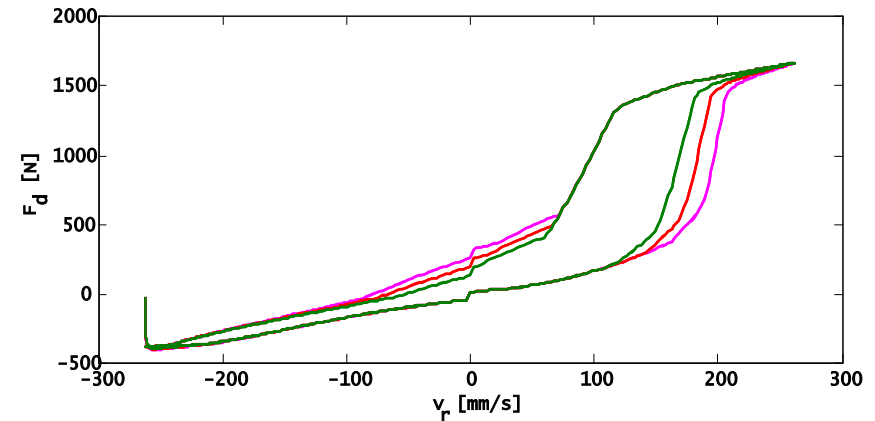


◆ Simulation results – FDV piston diameter, piston travel

Piston diameter ( $D_p$ )



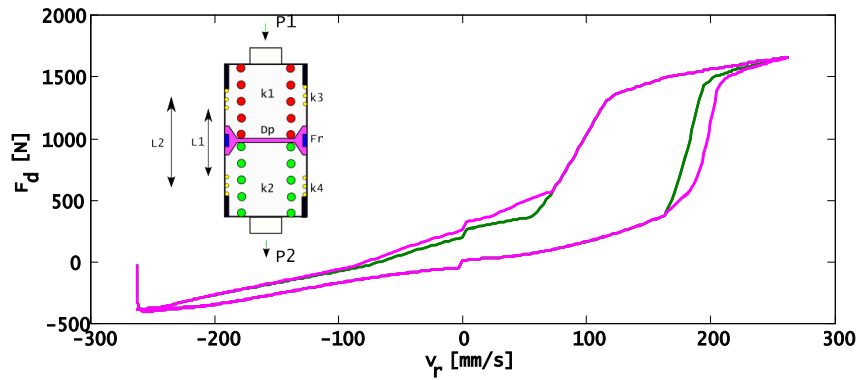
Piston travel ( $L_1$ )



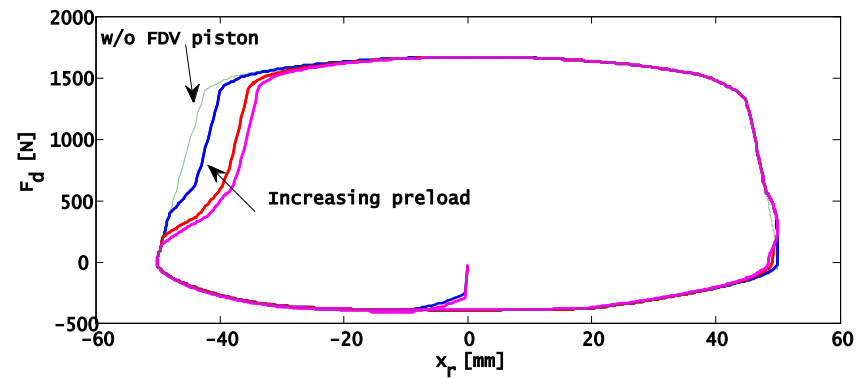
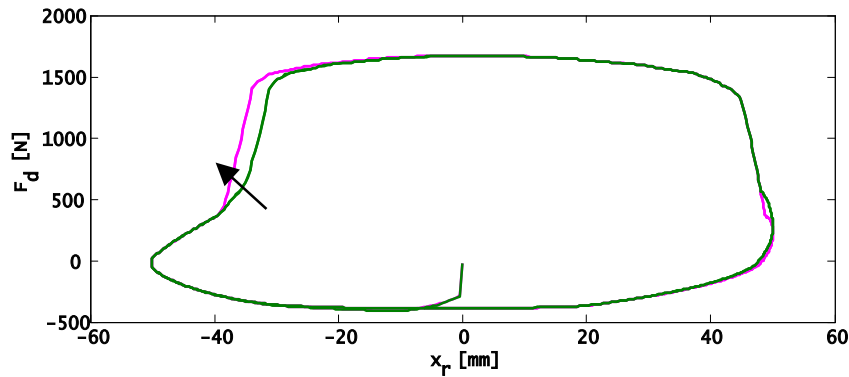
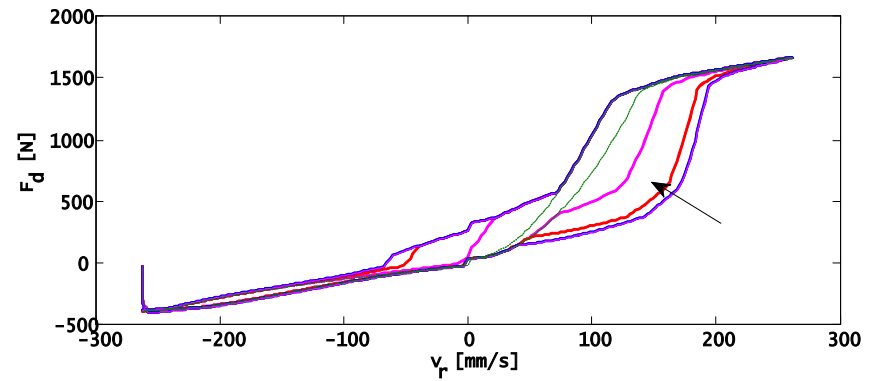


◆ Simulation results – Cushioning springs, valve preload

Cushioning springs ( $k_3, k_4$ )



Preload change (entry/exit blow-offs)



## ◆ Damper modeling tools

- Damper modeling tools provide valuable insight into damper physics and interactions between its key components
- The models form basis for simulating and developing more advanced passive & semi-active damping technologies

## ◆ Frequency-Dependent Valve (FDV) valve

- FDV valves are a simple yet effective way of enhancing the performance of a passive shock absorber @ standard and elevated frequencies
- Developing the model of an FDV valve was necessary to understand the influence design variables have on the valve performance
- Preliminary vehicle studies have shown well-designed FDVs may contribute to better isolation from road inputs at frequencies above the wheel resonant frequency w/o any or little degradation of ride and handling metrics