

An advanced physical modeling approach for brake system performance analysis



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Brake system modeling for vehicle performance evaluations

- I. Project Context and Positioning
- II. Modeling Approach and Technical Details
- III. Simulated maneuvers in normal and emergency conditions
- IV. Simulated maneuvers with system failures
- V. Conclusions and possible model extensions

Presentation: Table of Content

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APG - Presentation of the Company

APG, Zhejiang Asia Pacific Mechanical & Electronic Company

- One of the major supplier of Brake Systems in China.
- This supplier is specialized in research, development & manufacturing of:

- Brake disc/drum assemblies
- Vacuum pump
- Master Cylinder
- Brake Booster
- ABS/ASR systems.
- ESP systems

Annual
production
capacity of 100K
sets

Historical
View



- Main customers: FAW-Volkswagen, Dongfeng-PSA, Chery, Chang'an Automotive,...

APG, Zhejiang Asia Pacific Mechanical & Electronic Company

- The current R&D process of the company is mainly based on **intensive test campaigns**
- APG is now looking for new R&D processes relying on simulation:
 - Improve the understanding of their systems
 - Decrease the testing effort
 - Accelerate development time/reduce time to production
 - Assist the development and validation of electronic controllers such as ABS/ASR/ESP control units

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Brake system modeling for vehicle performance evaluations

1. Component modeling and behavior validation (one after one)

- Dual Brake Booster
- Master Cylinder
- Front floating Caliper
- Rear floating Caliper with park brake assembly



- ### 2. Hydraulic piping & hoses: modeling according to the frequency range of interest
- ### 3. Complete Brake System assembly: parameter adaptation and validation
- ### 4. Final validation of the Brake System Modeling Platform

Component Modeling and Unitary Validations

Vacuum Dual Brake Booster

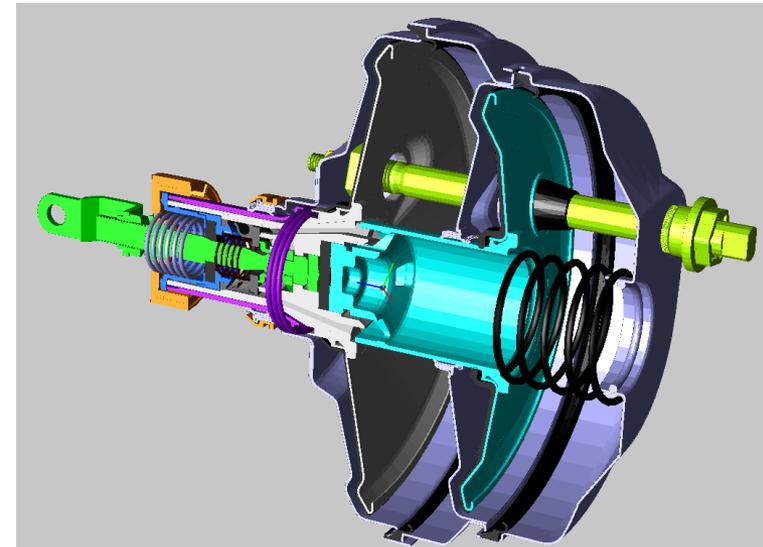
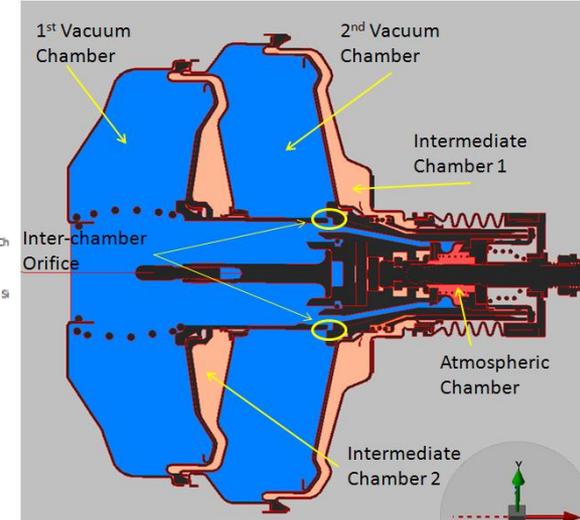
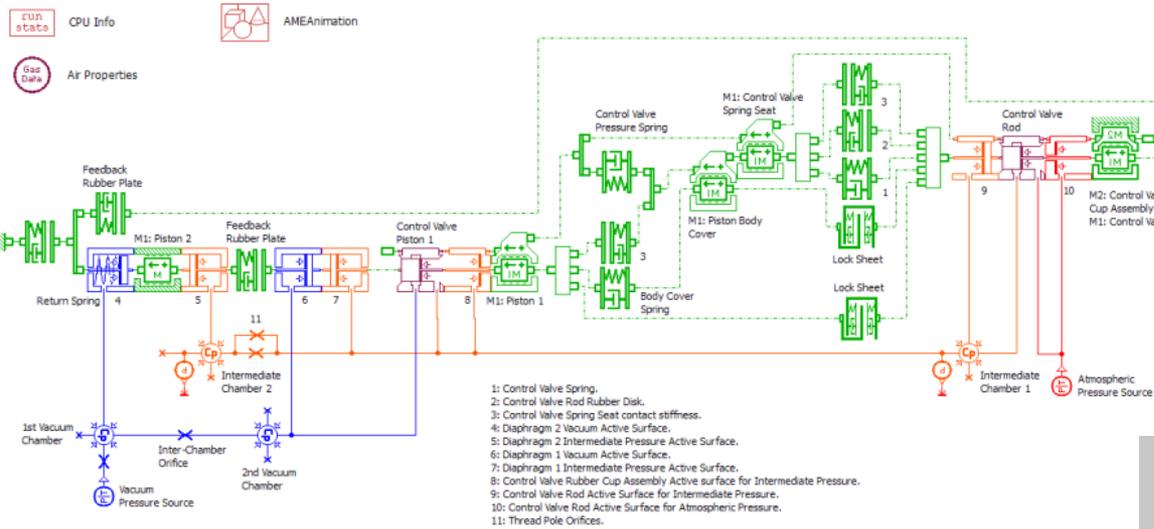
Date: June 2010
Author: NSR, LVE



APG BRAKING SYSTEM : BOOSTER MODEL



LMS-Imagine
Engineering Services
Lyon, France



✓ Functional springs & body relative endstops (Mechanical Domain).

✓ Vacuum vs. atmo. pressure: functional assistance with typical active surfaces (Pneumatic domain)

Component Unitary Validation

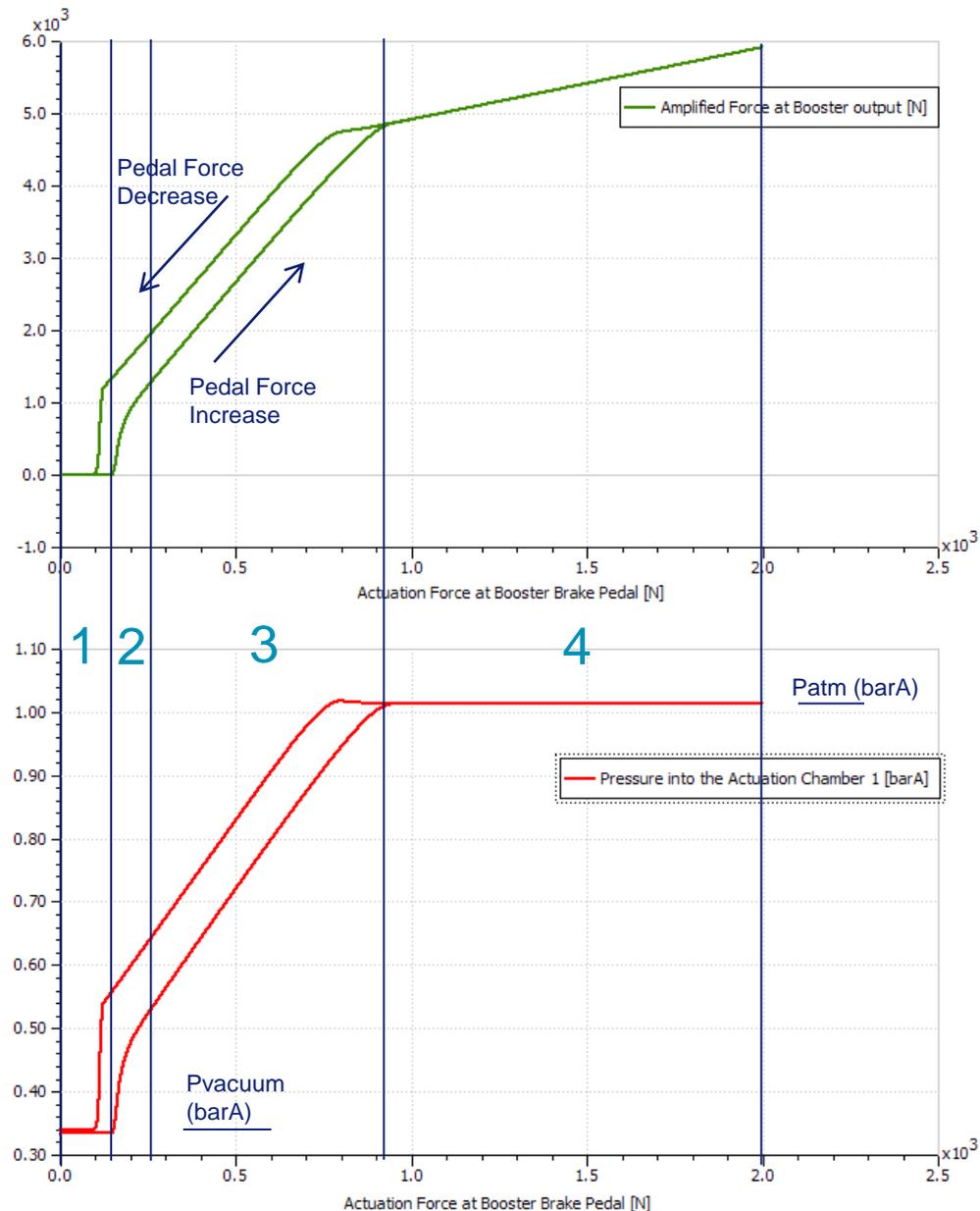
■ Vacuum Dual Brake Booster

The booster behaves according to 4 main phases:

1. Spring preloads overcome
2. Closing of valves with contact flexibilities
3. Amplification due to vacuum assistance
4. Saturated assistance →
ratio = input/output = 1

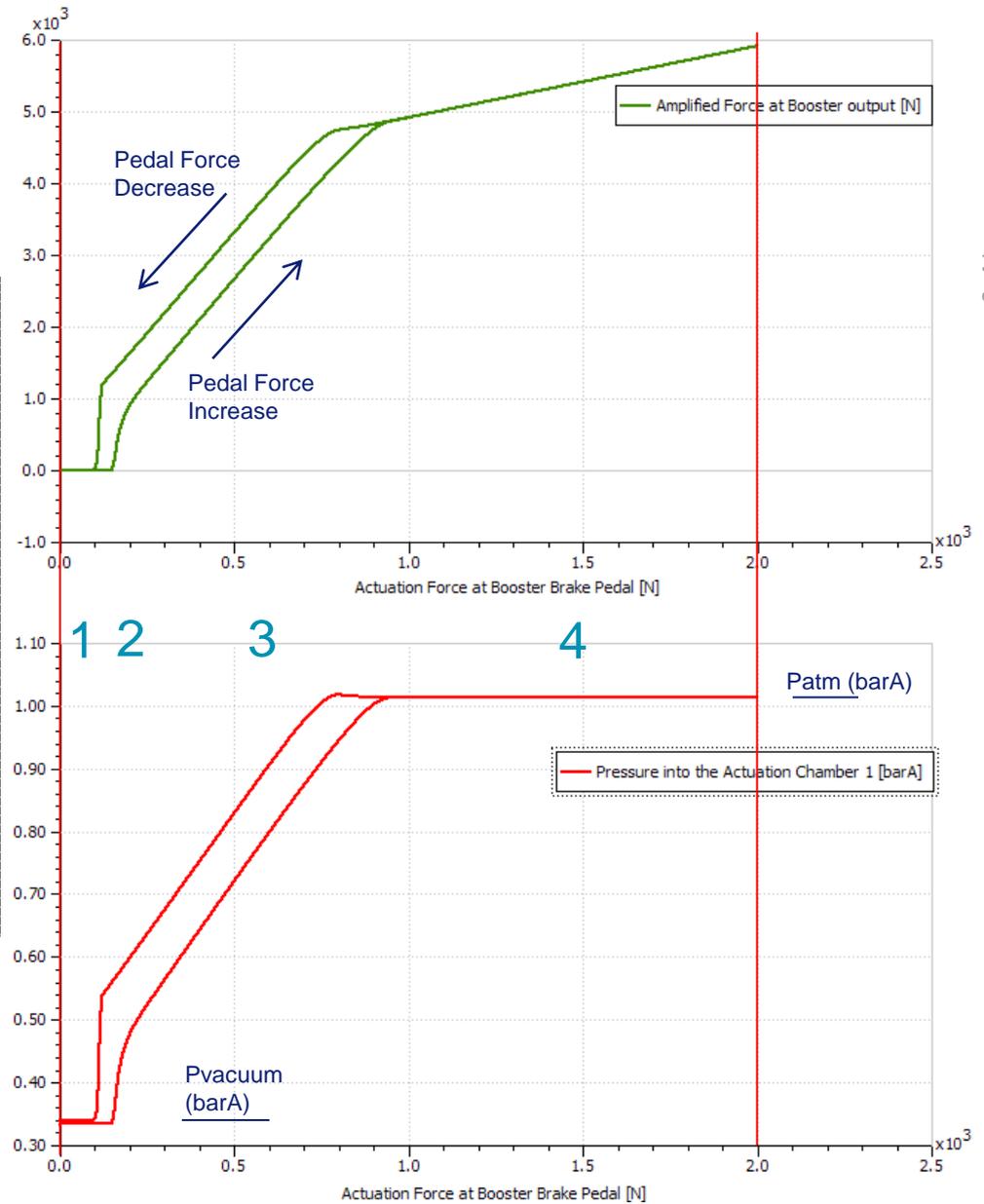
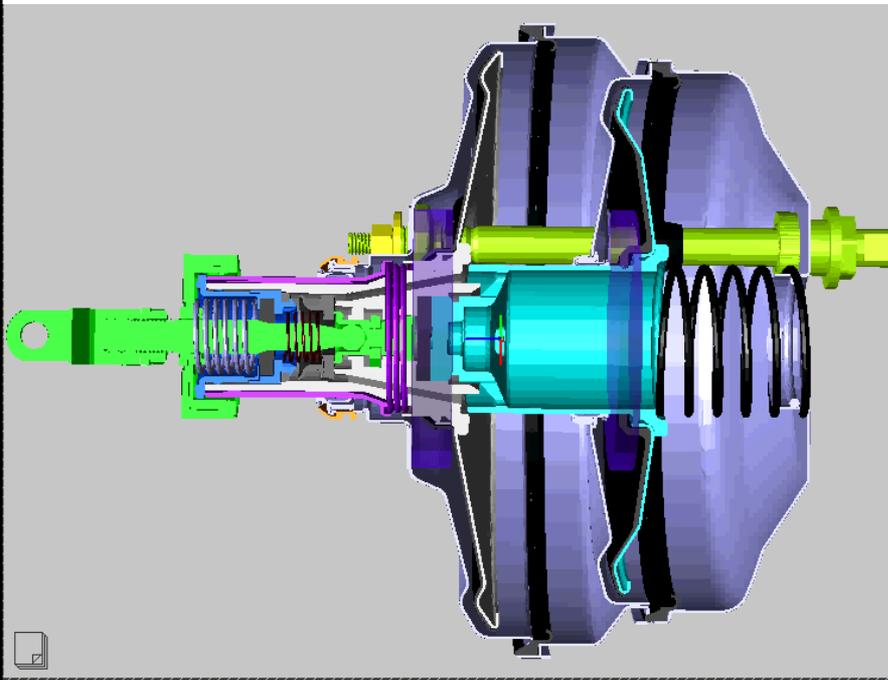
Boundaries: “Quasi-Static Actuation”

→ Validation of booster alone
with an Artificial Master Cylinder modeled
by means of an equivalent stiffness



Component Animation

■ Vacuum Dual Brake Booster



Component Modeling and Unitary Validations

■ Master-Cylinder

Date: March 2010
Author: NSR, LVE



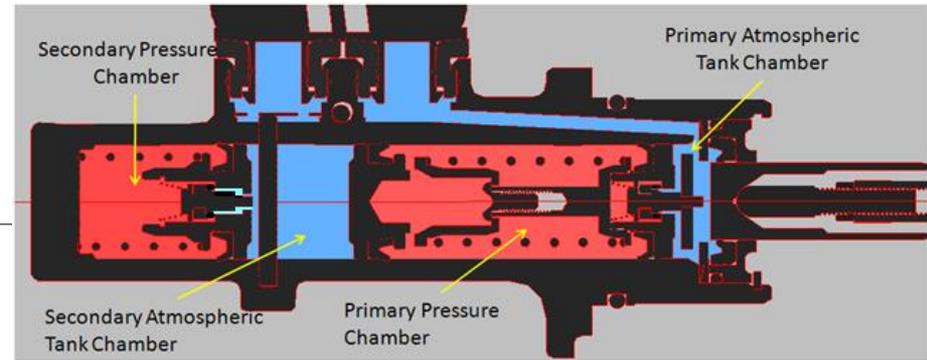
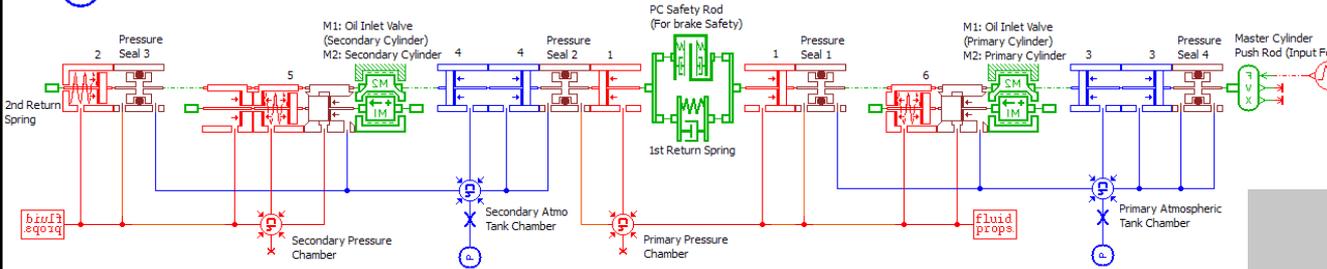
APG Braking System: Master Cylinder Model

LMS-Imagine
Engineering Services
Lyon, France

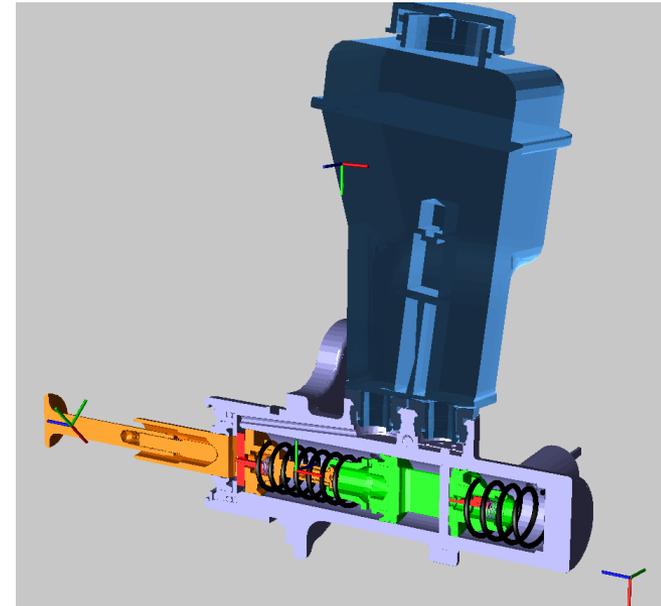
CPU Time Info

AMEAnimation

Liquid Props



- ✓ Functional springs & body relative endstops (**Mechanical domain**).
- ✓ Brake vs. atmo. pressure: functional primary & secondary chambers with typical active surfaces (**Hydraulic domain**)

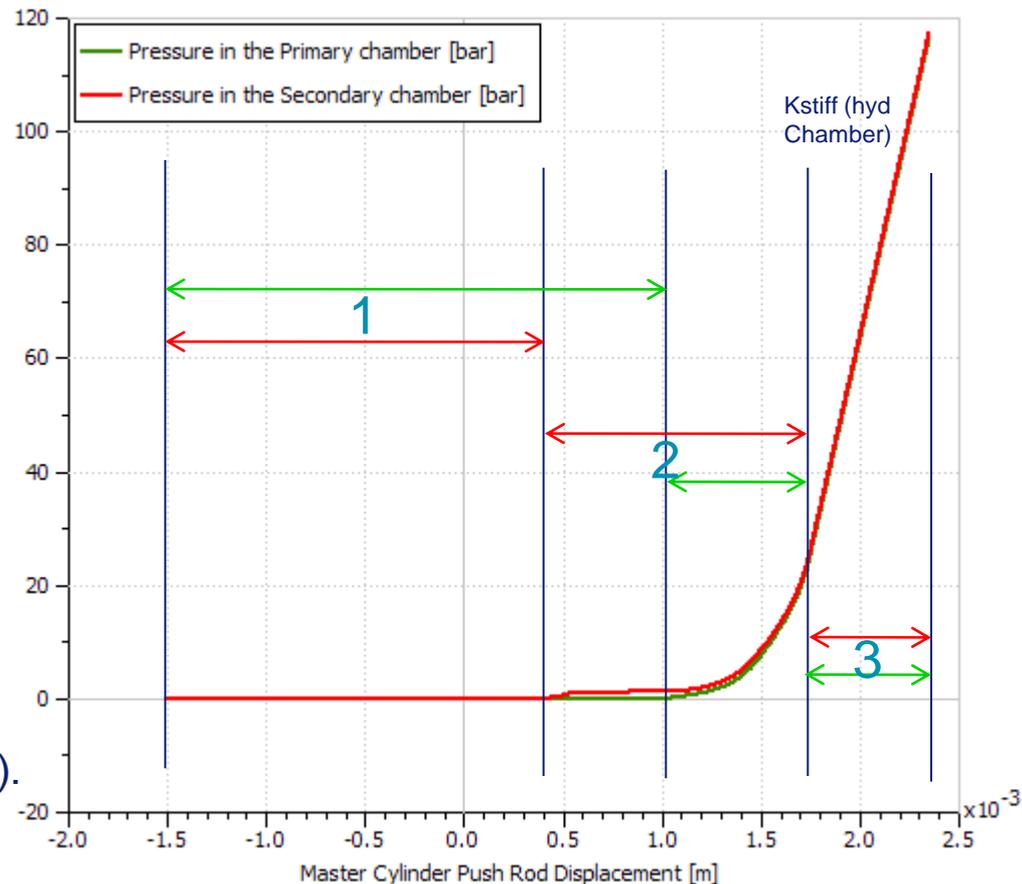


Component Modeling and Unitary Validations

■ Master-Cylinder

The Master-Cyl. behaves according to 4 main phases:

1. Dead stroke (Valves are opened).
2. → Closing of 2nd and 1st valve with contact flexibilities.
→ Fluid compression and air gas content dissolution (quadratic shape).
3. Pressure rise up according to hydraulic closed volume and spring loads (linear behavior “Kstiff”).

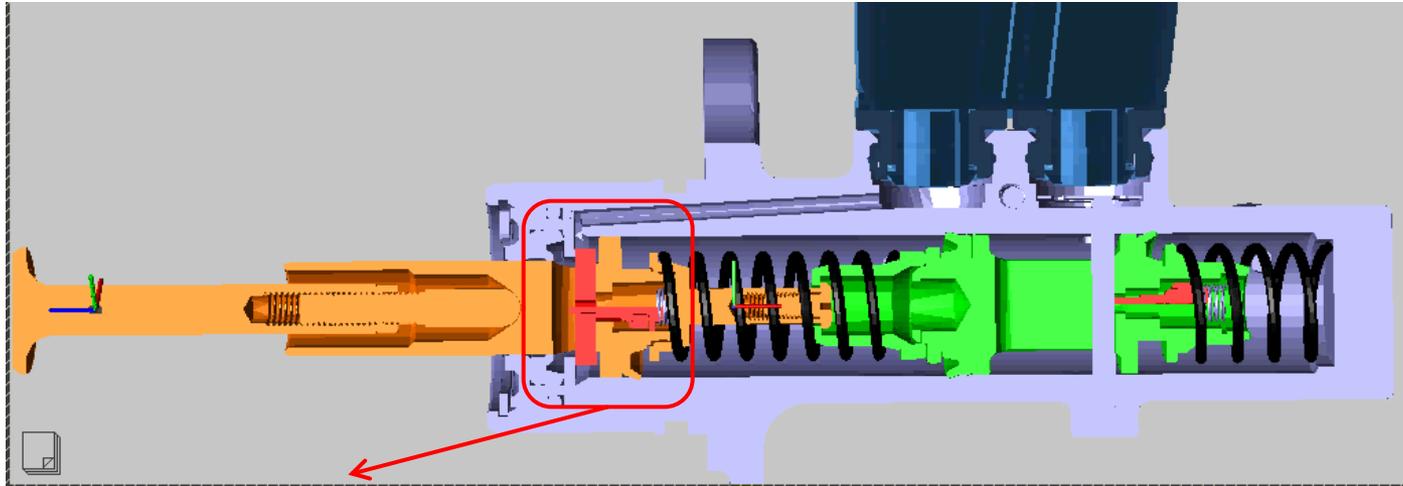


Boundaries: “Quasi-Static Actuation”

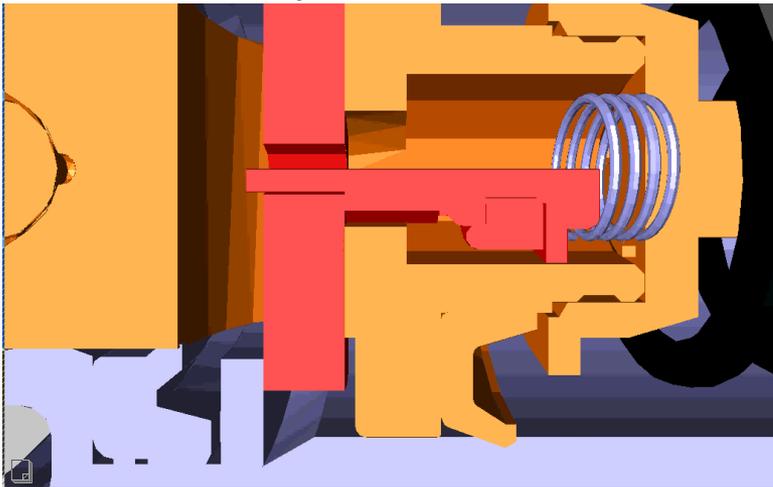
→ 1st and 2nd Hydraulic chambers outlets in closed condition

Component Animation

- **Master-Cylinder:** → Behaviors of 1st Valve (Primary Chamber) and 2nd Valve (Secondary Chamber)



Primary Chamber Valve



Component Modeling and Unitary Validations

Front Floating Caliper

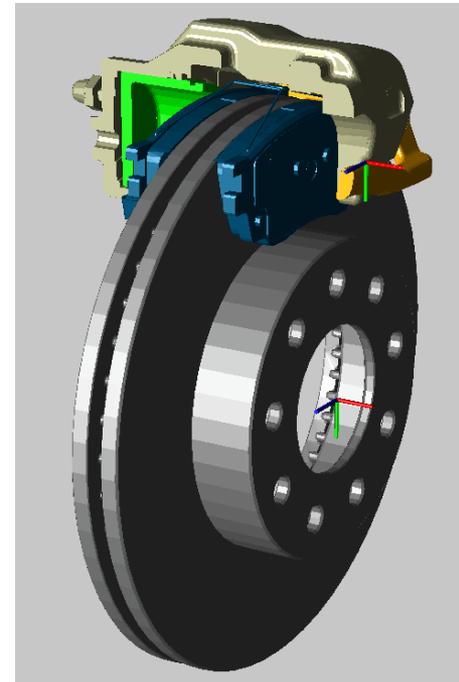
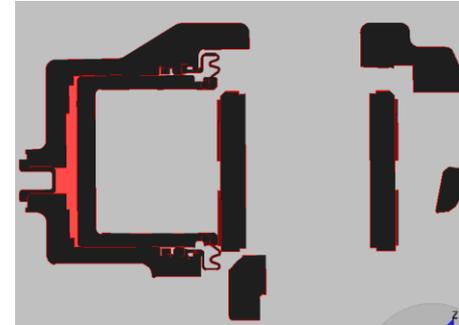
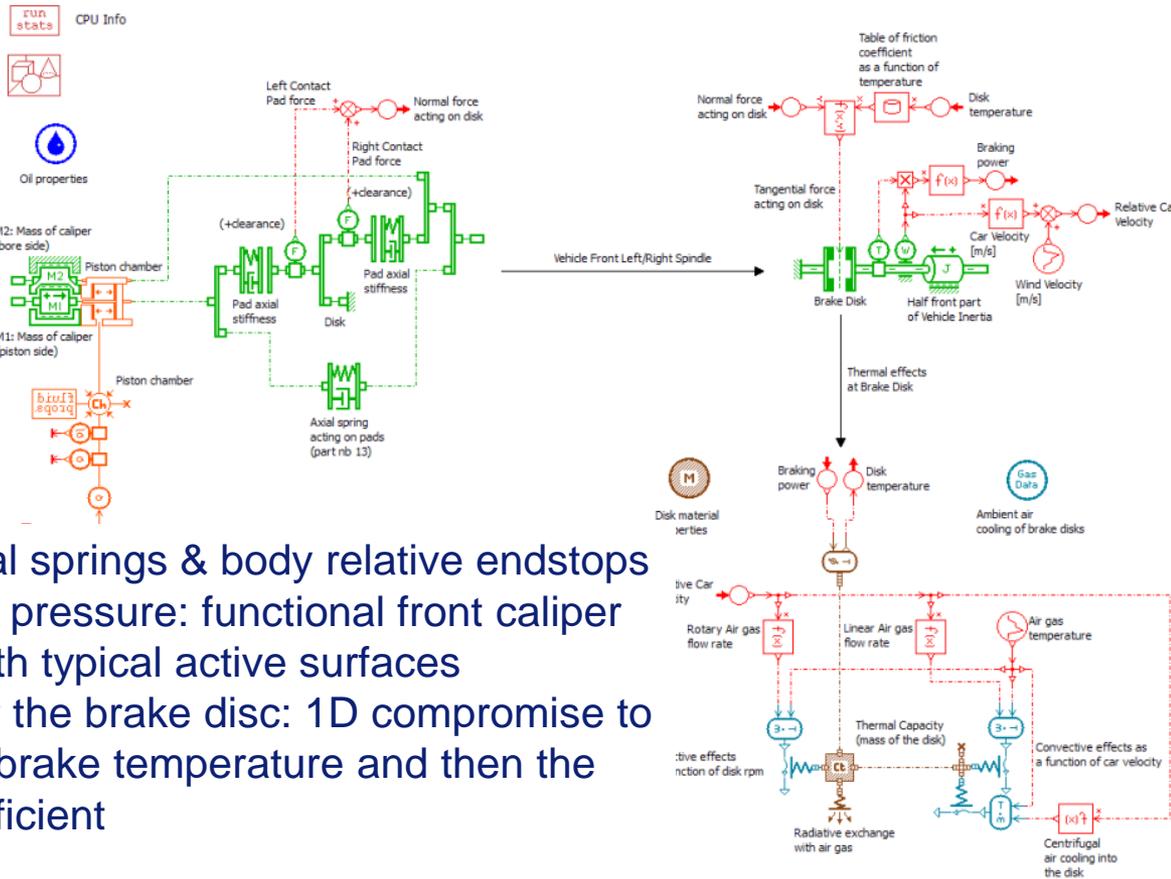
Date: April 2010
Author: NSR, LVE



APG BRAKING SYSTEM : Front Brake Disk Model



LMS-Imagine
Engineering Services
Lyon, France



- ✓ Functional springs & body relative endstops
- ✓ Hydraulic pressure: functional front caliper chamber with typical active surfaces
- ✓ Cooling of the brake disc: 1D compromise to identify the brake temperature and then the friction coefficient

Component Modeling and Unitary Validations

Rear Floating Caliper with park brake system

Date: June 2010
Author: NSR, LVE



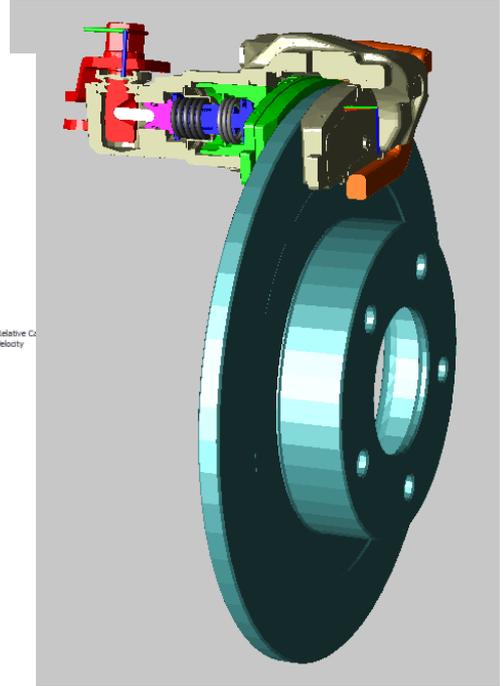
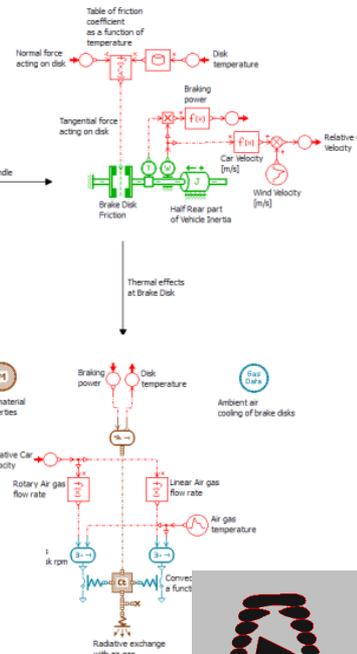
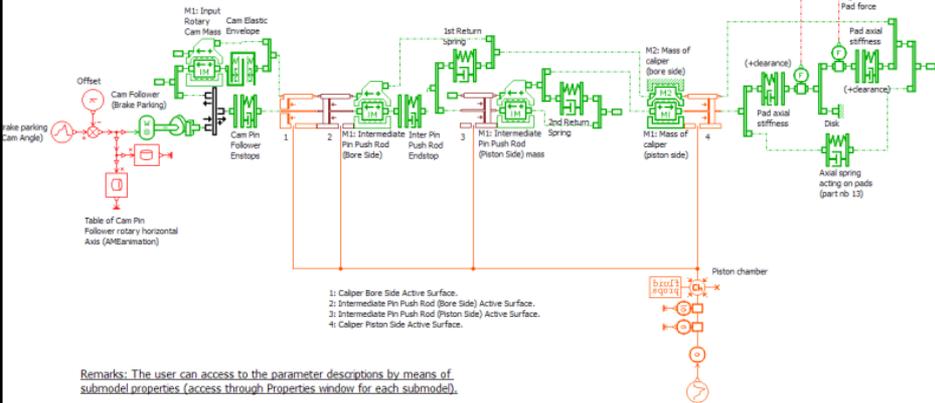
APG BRAKING SYSTEM : Rear Brake Disk Model



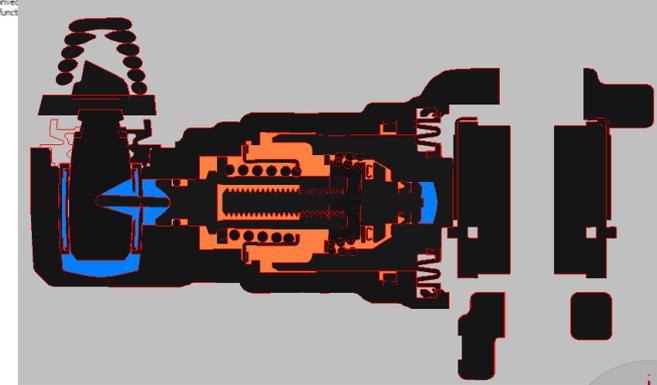
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OPU Info AMExamination

Oil properties

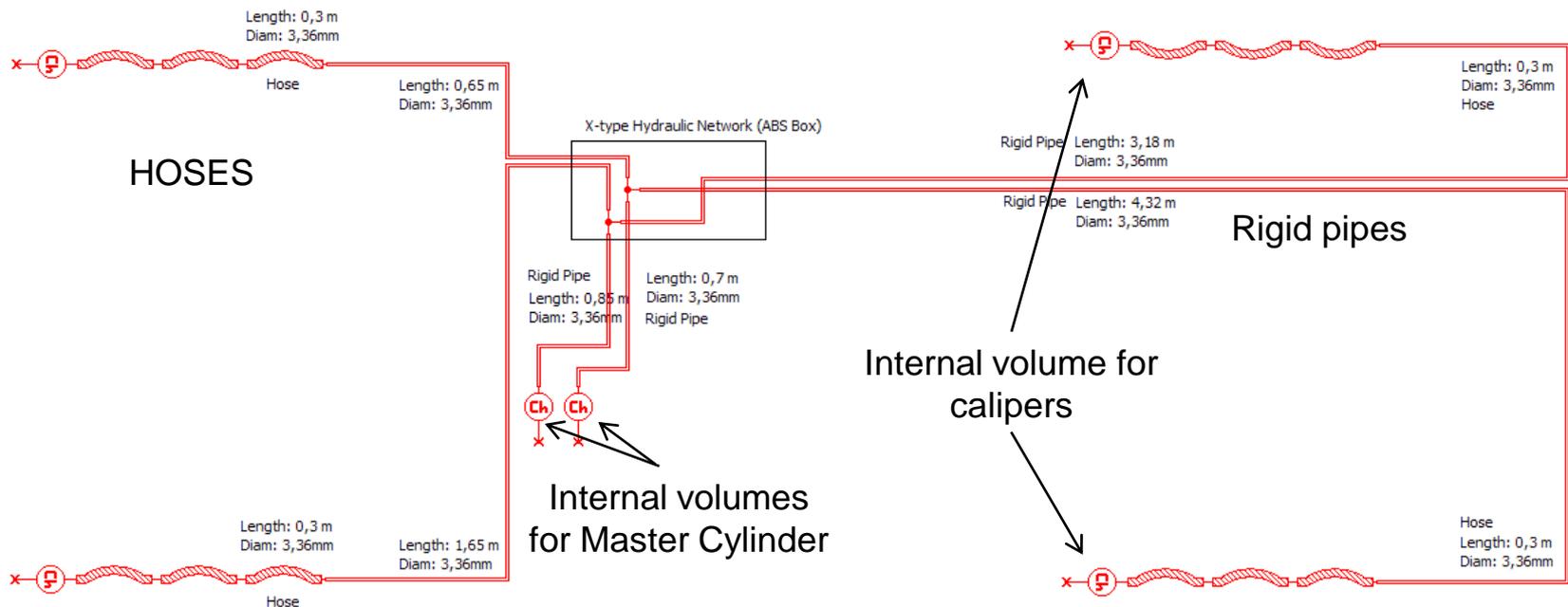


- ✓ Spring & body relative endstops, Cam Follower for the park brake system
- ✓ Hydraulic pressure: functional rear caliper chamber with typical active surfaces
- ✓ Cooling of the brake disc: 1D compromise to identify the brake temperature and then the friction coefficient



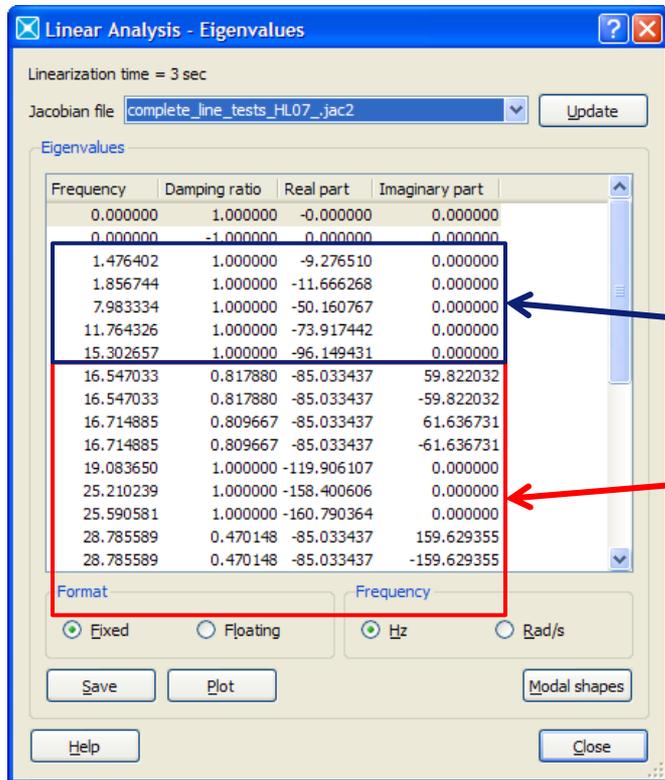
Hydraulic Piping Modeling

- **Identification of the targeted frequency range for the braking application:**
 - Our model does not include ABS/ESP system which can imply fast dynamics. Hence, no need to go above the maximum excitation frequency that the driver can generate at the brake pedal.
 - In agreement with APG engineers, we assumed that the driver is able to apply at the brake pedal up to 3 Hz excitation.
 - Then, the frequency range of interest was 0-3 Hz for the brake model.



Hydraulic Piping Modeling

- Methodology for selection of the appropriate lines & hoses:
 - ✓ Eigenvalues of the Hydraulic Network with HL07 (pipes, IR-C) and HH04R (hoses, same level C-IR):



Time Constant:
1st order lag due
to RC elements

pipng modes:
from 16 Hz up to
160 Hz

Conclusions:

the line & hose natural frequencies begin at 16 Hz. The frequency range of interest for our model is below this frequency (3 Hz).

We can decrease the modeling complexity and remove the line modes being not excited. HL07 and HH04R are replaced with HL01 lines (RC elements)

Brake system modeling for vehicle performance evaluations

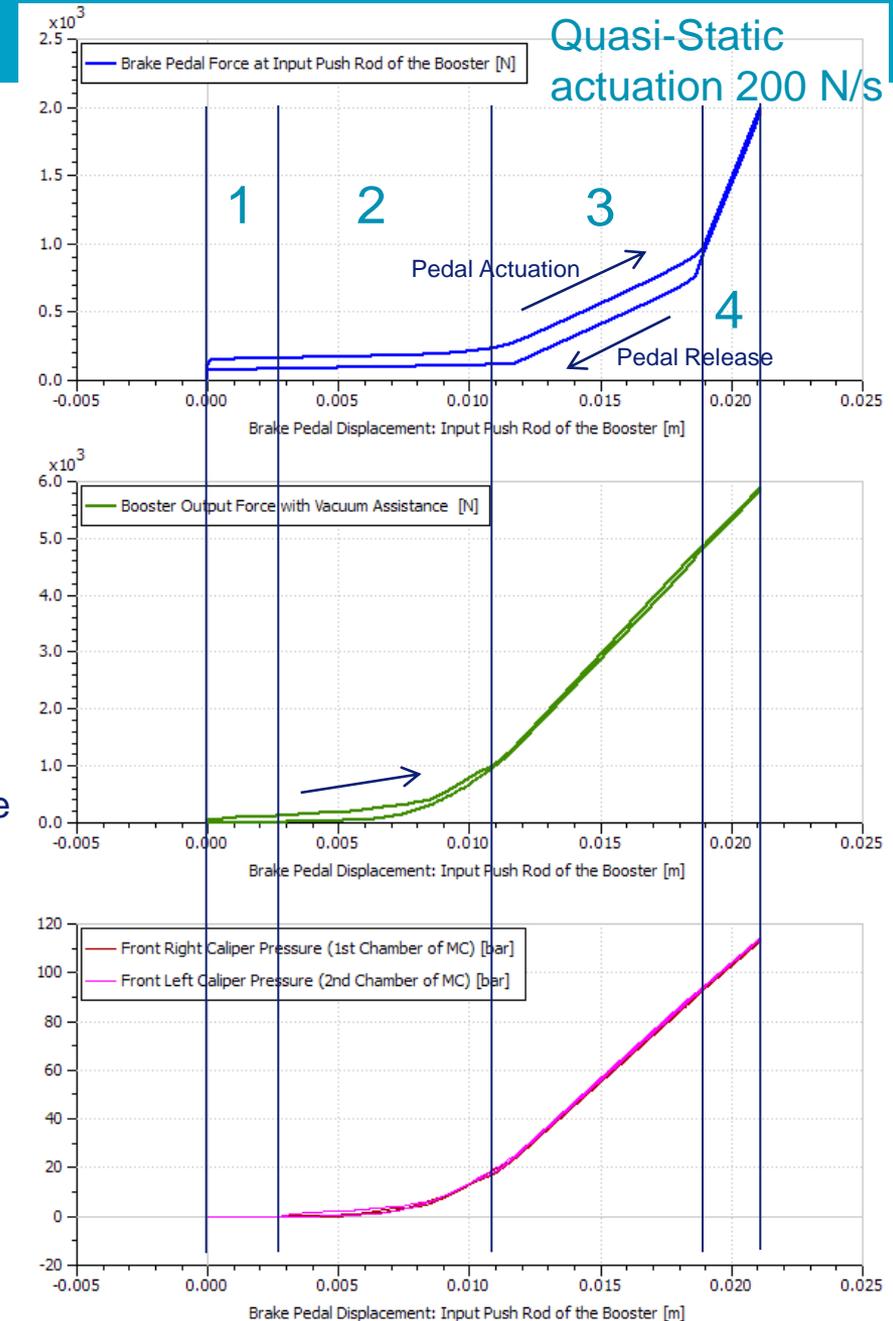
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Brake Pedal Feel

▪ Booster and Master Cylinder Assembly + Calipers and Hydraulic Hoses & Pippings

The Driver Brake Pedal Feel is mainly conditioned by the here below aspects:

1. Pedal Dead Stroke (non reaction travel)
→ Due to clearances before closing of Valves in the Master Cylinder.
2. Pedal Pumping Stroke (low reaction travel)
→ Air gas content dissolution contained into the brake and valve contacts deflection in the booster.
3. Pedal Assisted Stroke (max reaction travel)
→ The Booster Vacuum Assistance is acting on the pedal amplifying the brake torque (caliper pressure)
4. Pedal Hard Stroke (regular reaction travel)
→ Saturated assistance in the booster



Brake Pedal Feel

■ Sensitivity of the Brake Pedal Feel as a function of Actuation Velocity

Batch Run Boundaries, Pedal Actuation:

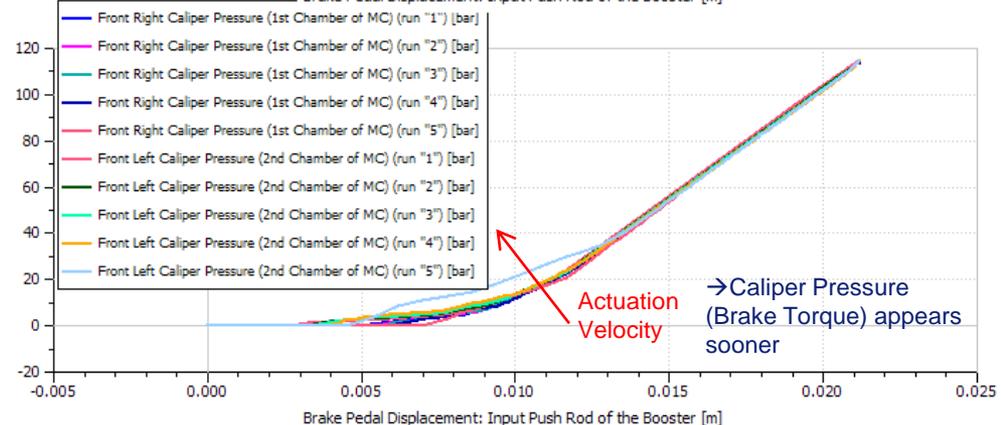
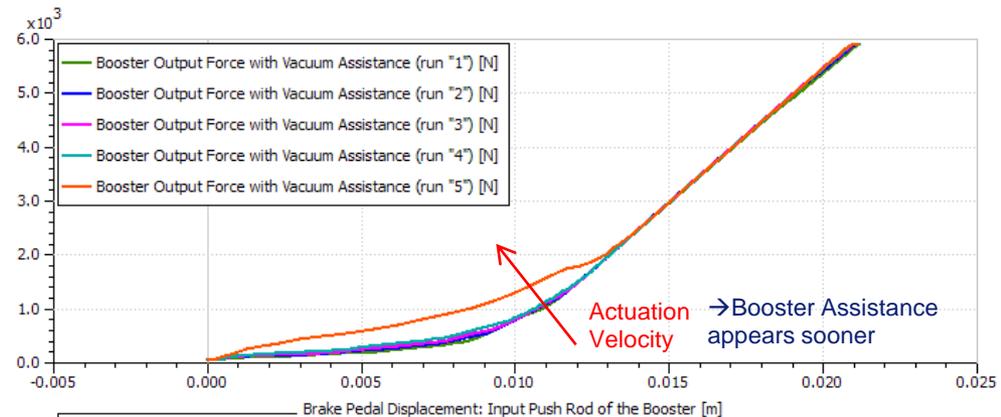
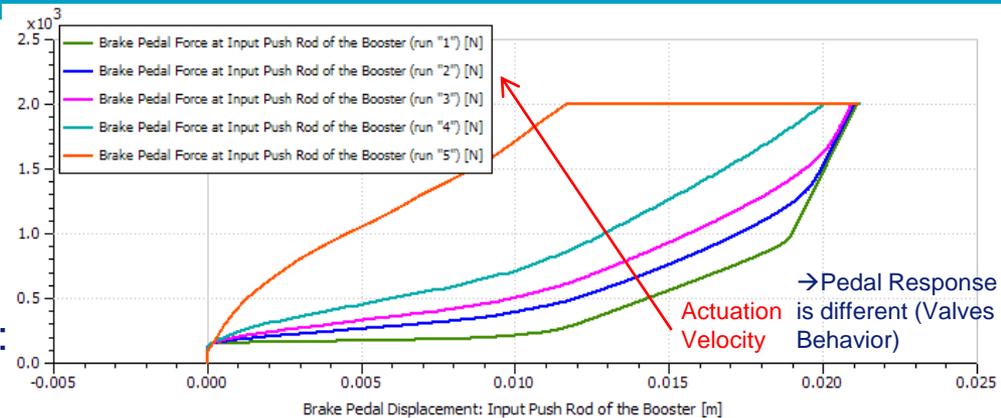
→ **Run 1: 22,2 N/s** (Quasi-Static Actuation: 2000 N in 90 sec).

→ **Run 2: 200 N/s** (Long brake Actuation: 2000 N in 10 sec).

→ **Run 3: 400 N/s** (Medium brake Actuation: 2000 N in 5 sec).

→ **Run 4: 1000 N/s** (Short and Strong Brake Actuation: 2000 N in 2 sec).

→ **Run 5: 10000 N/s** (Emergency Brake Actuation: 2000 N in 0,2 sec).

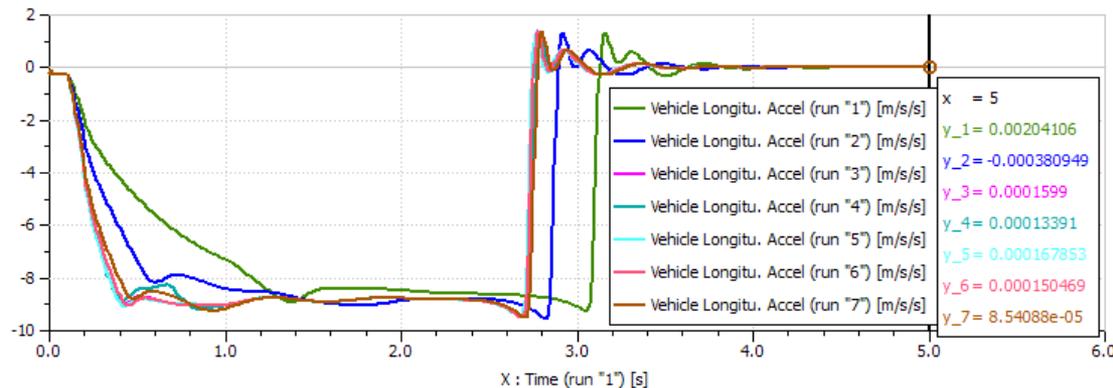
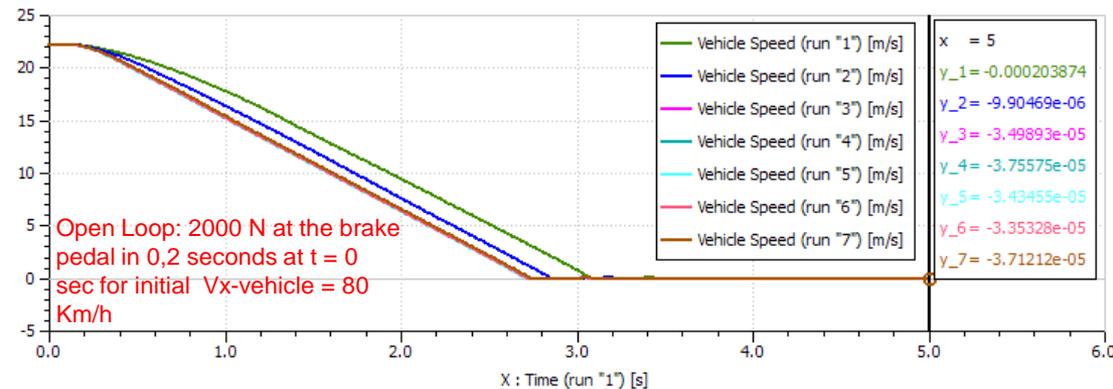
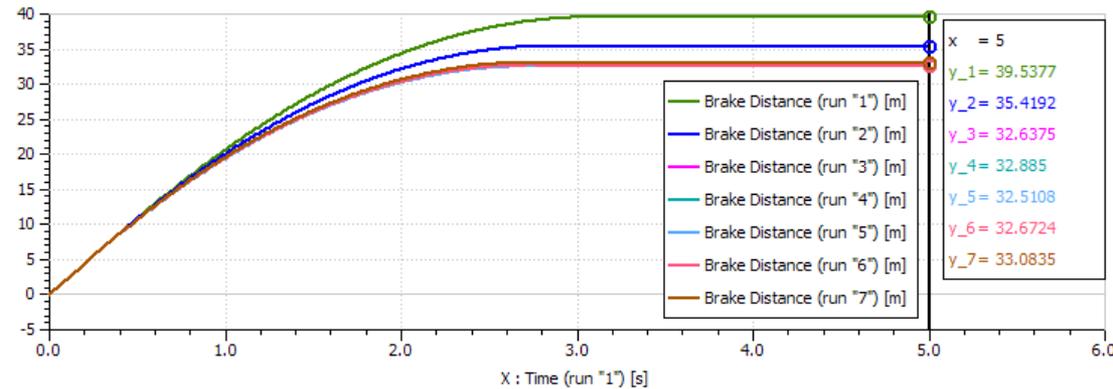


Brake Distance

Sensitivity of the Brake distance as a function of Brake Disk Temperature

Disk Temperatures:

- **Run 1:** Front Disks 20°C, Rear Disks 20°C.
- **Run 2:** Front Disks 50°C, Rear Disks 30°C.
- **Run 3:** Front Disks 80°C, Rear Disks 60°C.
- **Run 4:** Front Disks 110°C, Rear Disks 80°C.
- **Run 5:** Front Disks 140°C, Rear Disks 110°C.
- **Run 6:** Front Disks 200°C, Rear Disks 170°C.
- **Run 7:** Front Disks 300°C, Rear Disks 270°C.



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Brake Distance with System Failures

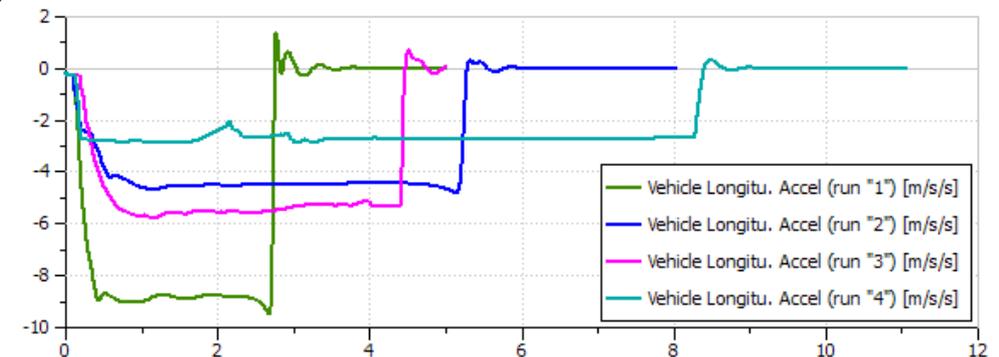
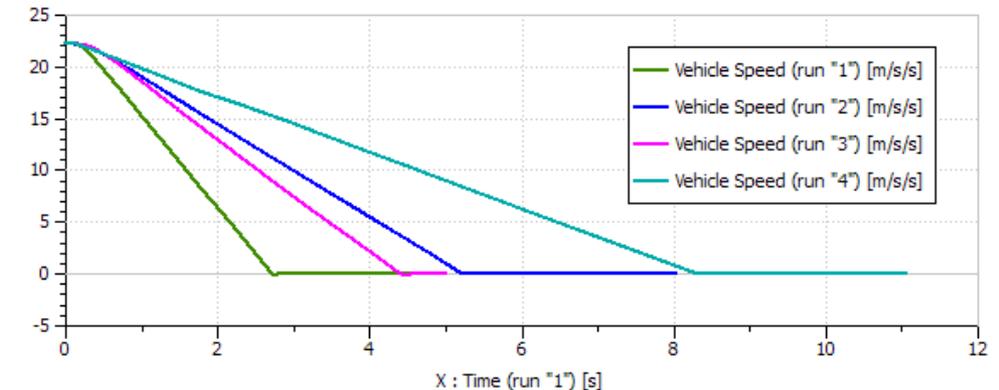
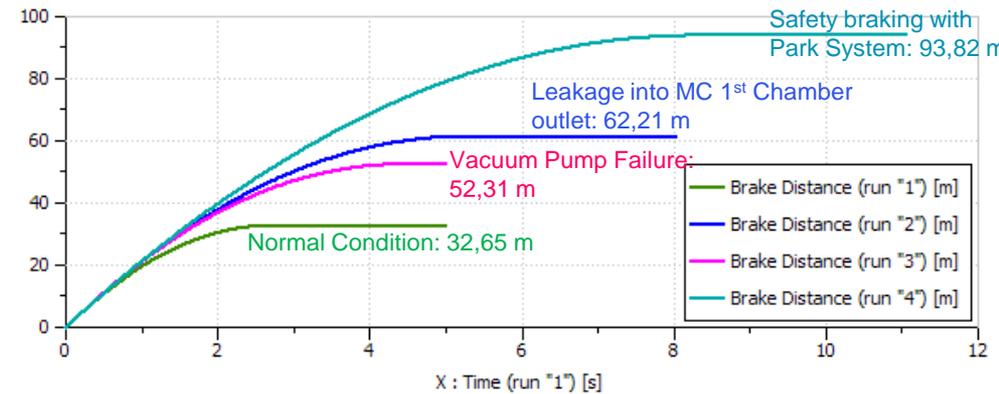
- What's happened in case of any sub-system failure ?

Simulated Failure Conditions:

- **Run 1:** Normal Conditions.
- **Run 2:** Brake Fluid Leakage at outlet of MC 1st Chamber.
- **Run 3:** Failure of Vacuum Assistance.
- **Run 4:** Braking by means of Parking System (Leakage at outlets of 1st and 2nd Chamber of Master Cylinder).

Boundaries: Open Loop (in terms of Lslip)

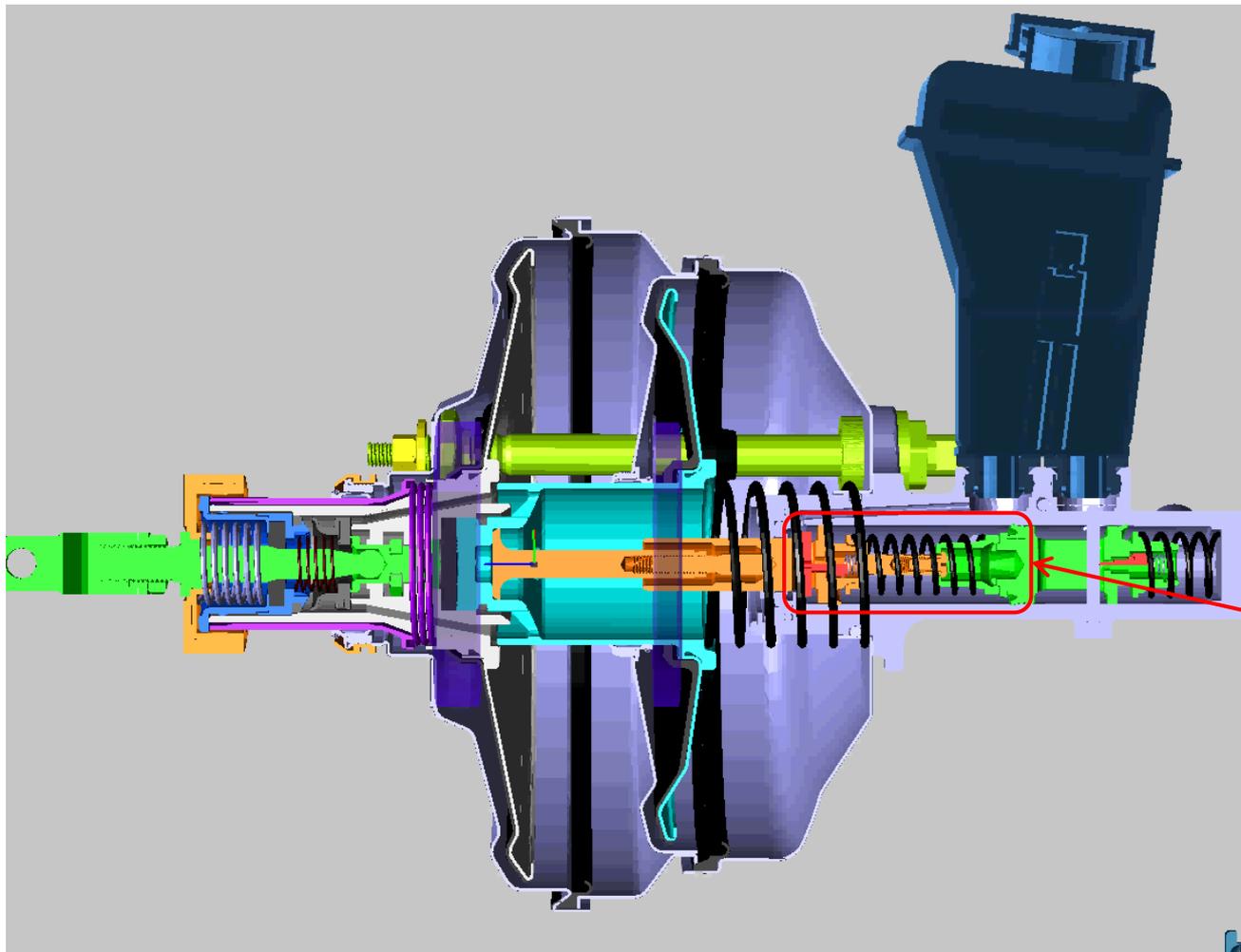
- ✓ 2000 N at the Brake Pedal in 0,2 second with initial vehicle Speed of 80 Km/h.
- ✓ Front Disk Temperatures: 80 °C.
- ✓ Rear Disk Temperatures: 60 °C.



Brake Distance with System Failures

- What's happened in case of any Sub-system failure ? AMEanimation View

Simulated Failure Conditions: → Run 2: Brake Fluid Leakage at outlet of MC 1st Chamber.



1st Chamber of Master Cylinder:

→ Leakage of Brake Fluid at piping outlet

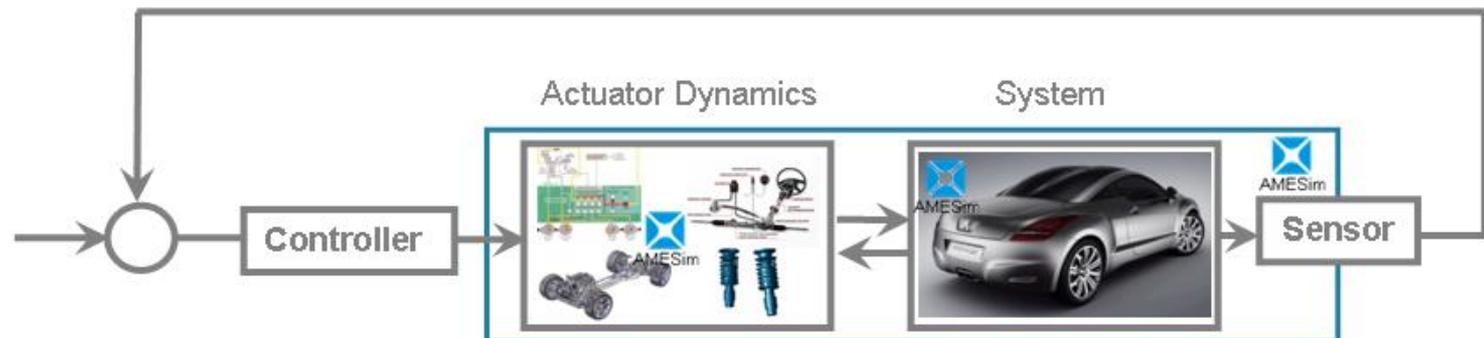
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Conclusions & Synthesis

■ AMESim Added Values:

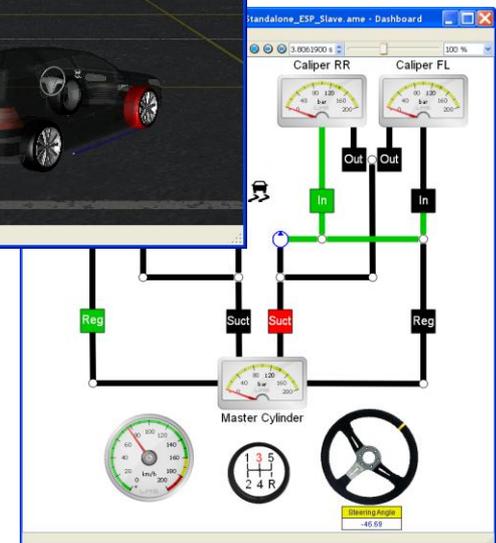
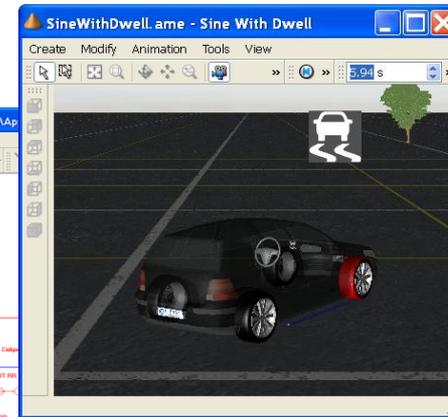
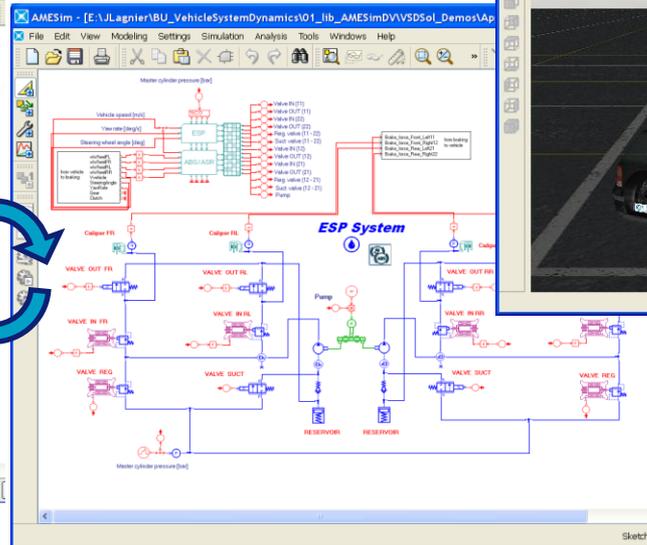
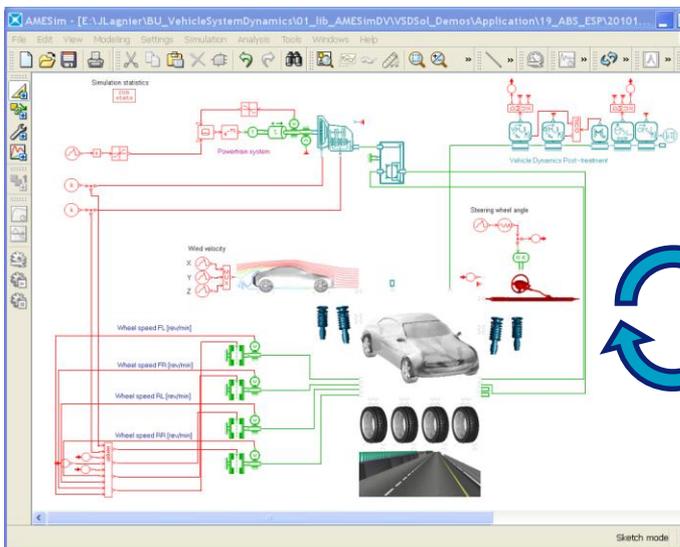
- The model combines **Mechanical** + **Hydraulic** + **Pneumatic** + **Thermal** phenomena libraries in a single platform.
- Scalability in the modeling assumptions.
- The AMEAnimation feature allows animating the CAD of the components in order to get a bit more insights into the booster, master cylinder and caliper behaviors.
- Even if the system does not include any “active” component, the solution is open to include an hydraulic block from an ABS/ESP and an interface to the control law.

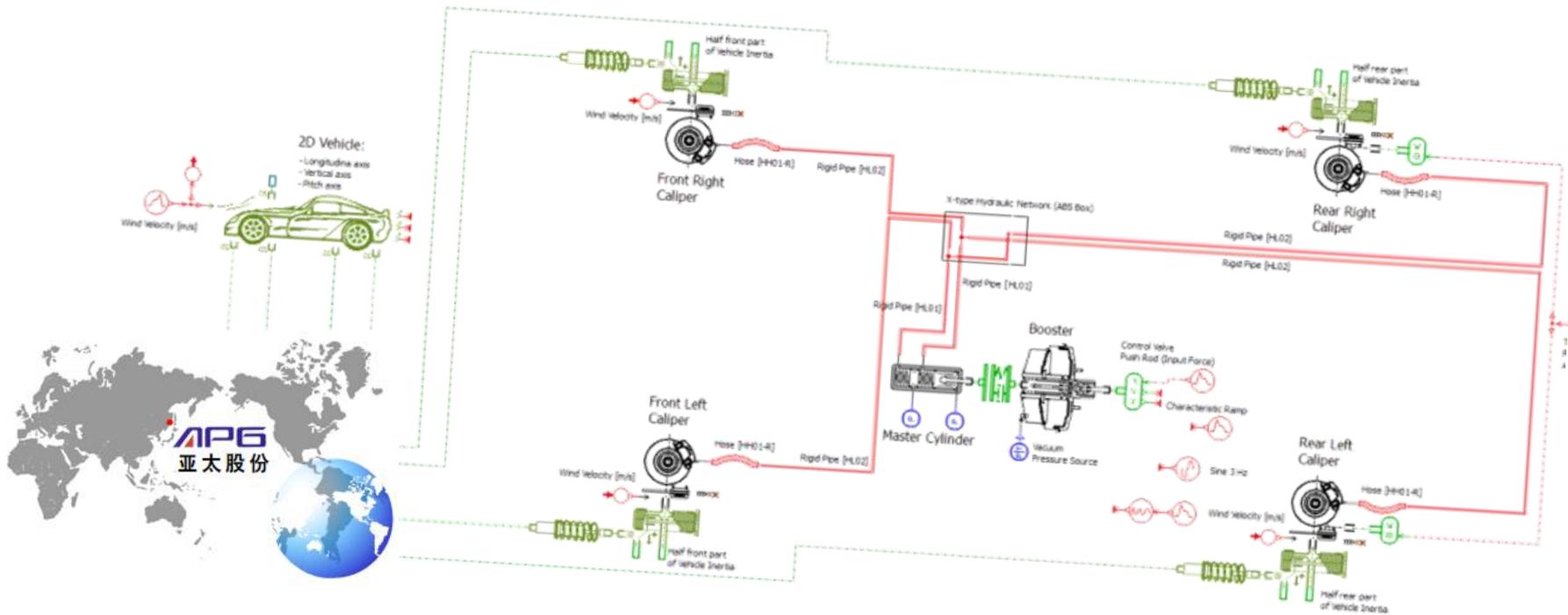


Future Extensions

■ APG Future ... involving ABS-ESP:

- The Hydraulic Component Design Library allows the design and analysis of any valve technology involved into ABS/ESP hydraulic block (HCU).
- AMESim includes special features for model simplification starting from the current model to reduce it in a way to be able to run it real time i.e. HiL testing.
- The Vehicle Dynamics library to take into account the 3D vehicle contributions when ABS/ESP is involved.





Thank you for your attention

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